

FINAL REPORT

on

DOUBLE TIME LAG COMBUSTION INSTABILITY MODEL
FOR BI-PROPELLANT ROCKET ENGINES

by

CHANG KENG LIU
Principal Investigator

Prepared for

National Aeronautics and Space Administration
George C. Marshall Space Flight Center
Marshall Space Flight Center, Alabama 35814

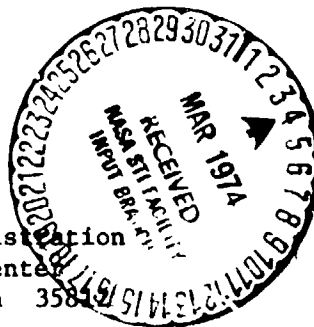
September, 1973

BER Report No. 158-80

(NASA-CH-120137) DOUBLE TIME LAG
COMBUSTION INSTABILITY MODEL FOR
BI-PROPELLANT ROCKET ENGINES Final Report
(Alabama Univ., University.) 45 p HC
\$5.25

CSCL 2CH 63/28 31664

674-18401



UA

FINAL REPORT

on

DOUBLE TIME LAG COMBUSTION INSTABILITY MODEL
FOR BIPROPELLANT ROCKET ENGINES

by

CHANG KENG LIU
Principal Investigator

Prepared for

National Aeronautics and Space Administration
George C. Marshall Space Flight Center
Marshall Space Flight Center, Alabama 35812

September, 1973

BER Report No. 158-80

DOUBLE TIME LAG COMBUSTION INSTABILITY MODEL
FOR BIPROPELLANT ROCKET ENGINES

INTRODUCTION

Since it was first observed in the early 1940's, low frequency combustion instability or chugging in liquid rocket engines has been the subject of many analyses. Von Karman (1) was the first to propose that the phenomenon is due to a combustion time delay between the instant of propellant injection and subsequent conversion into combustion products. In 1950 Gunder and Friant (2) presented an analysis in which this combustion delay was the essential feature but also included the inertia of the liquid in the feed system. This was followed about one year later with an analysis by Summerfield (1) which also incorporated the combustion delay and feed system inertia. Gunder and Friant treated both monopropellant and bipropellant rocket systems with a common combustion delay for the latter case. Summerfield treated only the monopropellant case. In both of these analyses the authors showed that instability is not possible if the pressure drop across the injector is greater than one-half the chamber pressure. Crocco and Cheng (3) later refined the model for the monopropellant case by assuming a time varying combustion delay which for simplicity was correlated to chamber pressure. Their analysis considered feed system inertance and capacitance as well as resistance. Hurrell (4) introduced the concept of an injection velocity-dependent combustion delay and

its effect on the neutral stability boundaries. More recently Wenzel and Szuch (5) conducted an analysis of the bipropellant case by allowing different vaporization rates for the two propellants. However, feed system inertance and capacitance were not considered. An interesting conclusion from this work was that in some cases decreasing the ratio of the injection pressure drop to chamber pressure results in a transition from unstable to stable operation. This cannot be predicted from single combustion delay analyses.

All of these analyses have served to establish the current knowledge and understanding of low frequency combustion instability and have guided the prevention and/or elimination of the phenomenon in past rocket engine development work. However, in analyzing current high chamber pressure, bipropellant rocket engines with propellants possessing distinctly different vaporization rates (time lags), these analyses have certain shortcomings. First, the analysis of a bipropellant rocket engine with a monopropellant model applied individually to each propellant system yields questionable results because it neglects the influence of the other system on the overall stability. Second, the use of a bipropellant model with a common combustion delay to represent propellants with distinctly different vaporization rates is unrealistic. And third, in view of the fact that feed system inertance and capacitance play an important role along with injection pressure drop in determining the stability of the overall system, these factors should also be included in the model. It should be possible, at least in some cases, to stabilize the combustion with less impact to the overall rocket system by optimizing these parameters rather than adjusting only the pressure drop. Reliance on an injection pressure drop greater than one-half the

chamber pressure to guarantee stability is unrealistic from both the standpoint of high pump discharge pressures that would be required for today's engines and from the results of the analysis of Wenzel and Szuch.

This report advances a bipropellant stability model in which feed system inertance and capacitance are treated along with injection pressure drop and distinctly different propellant time lags. The model is essentially an extension of Crocco's and Cheng's monopropellant model to the bipropellant case assuming that the feed system inertance and capacitance along with the resistance are located at the injector. The neutral stability boundaries are computed in terms of these parameters to demonstrate the interaction among them.

Combustion Chamber and Associated Equations

Crocco's ingenious derivation of the combustion chamber equation is based only on a limited number of fundamental assumptions and definitions. First, like Summerfield, he postulates that the chamber pressure p is a function of time t , even when the combustion process is a steady one. Next, he suggests that the rate of combustion processes in the chamber is a function of several variables, the two more prominent of which are pressure and temperature; the rest of the relevant variables are lumped into a single group, Z . Thus, without identifying the nature of the combustion process, he states neatly,

$$\text{Process Rate} = f(p, T, Z) = f(\bar{p}, \bar{T}, \bar{Z}) \left[1 + p' \frac{\text{H.D.T.}}{f(\bar{p}, \bar{T}, \bar{Z})} \right] \quad (1)$$

where the bars on the variables indicate the steady-state values of these variables, and $p' = p - \bar{p}$ is the small perturbation of the chamber pressure, and H.D.T. denotes the higher derivative terms, such as

$$\left(\frac{\partial f}{\partial p} + \frac{\partial f}{\partial T} \frac{dT}{dp} + \frac{\partial f}{\partial Z} \frac{dZ}{dp} \right) \bigg|_{p = \bar{p}, T = \bar{T}, Z = \bar{Z}}.$$

In so doing, Crocco singles out the predominant effects of pressure on the process rate f , and disregards those of the others. As it will be shown later that, out of this perfectly general and vague definition of f , Crocco was able to lay the foundation for the formulation of the relationship between the burning and the injection rates of the propellant.

From the definition of f just introduced, it is easily obtained by transposition

$$n = \frac{p'(H.D.T.)}{f(\bar{p}, \bar{T}, \bar{Z})} = \frac{f(p, T, Z) - f(\bar{p}, \bar{T}, \bar{Z})}{f(\bar{p}, \bar{T}, \bar{Z})} \quad (2)$$

where n is simply the percent change of process rate, with respect to the steady state process rate, due to a small perturbation in pressure, p' . Crocco and Cheng call this % change the interaction index. This index seems to reflect, to some degree, the design of combustion chamber.

The next innovation that Crocco introduced at the outset of his investigation in combustion instability is the concept of a time lag, τ_t , which is the total time elapsed between the instant when $n = 0$ and the instant when $n = 1$. To simplify the ensuing analyses, he assumes further that during a certain portion of this time lag, i.e. τ_i , the interaction index n is zero and during the rest, i.e. $\tau = \tau_t - \tau_i$, $n = 1$. Thus, τ_i is the insensitive part and τ is the sensitive part of the total time lag τ_t .

With these two quantities defined, the following statement concerning the energy E_a contained in a certain element of propellant as it transforms from liquid state into gaseous products of combustion can be made

$$\int_{t-\tau}^t f(t') dt' = E_a \quad (3)$$

Note that the lower limit of the integral is the instant when the process rates begin to be affected by the combustion processes. Note, too, that this same level of energy E_a would have been reached also, if the processes of combustion had been steady. Thus, it is equally valid

$$\int_{t-\bar{\tau}}^t \bar{f}(t') dt' = E_a \quad (4)$$

where the schematic variation of the rate function f with respect to the time lags τ and $\bar{\tau}$ are shown in Figure A.

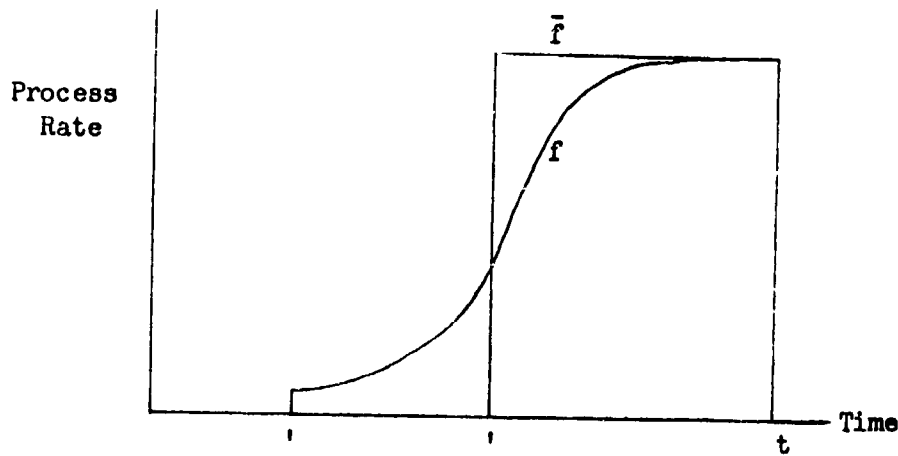


Fig. A

Observing the schematic variation of $f(t)$, one may write

$$\int_{t-\tau}^{t-\bar{\tau}} f(t') dt' + \int_{t-\bar{\tau}}^t f(t') dt' = E_a = \int_{t-\bar{\tau}}^t \bar{f}(t') dt' \quad (5)$$

The first integral on the left-hand side of (5) can be rewritten approximately,

$$\int_{t-\tau}^{t-\bar{\tau}} f(t') dt' \approx [(t - \bar{\tau}) - (t - \tau)] \bar{f}(t - \bar{\tau}) = (\tau - \bar{\tau}) \bar{f}(t - \bar{\tau}) \quad (6)$$

Transposing,

$$t - \bar{\tau} = \frac{1}{\bar{f}(t - \bar{\tau})} \int_{t - \tau}^{t - \bar{\tau}} f(t') dt' \quad (7)$$

But, from (5),

$$\int_{t - \tau}^{t - \bar{\tau}} f(t') dt' = - \int_{t - \bar{\tau}}^t [f(t') - \bar{f}(t')] dt'$$

hence, (7) becomes

$$\tau - \bar{\tau} = - \int_{t - \bar{\tau}}^t \frac{f(t') - \bar{f}(t')}{f(t - \bar{\tau})} dt' \sim - \int_{t - \bar{\tau}}^t \frac{f(t') - \bar{f}(t')}{\bar{f}(t')} dt'$$

Using the definition of n from (2), we have

$$\tau - \bar{\tau} = - \frac{n}{\bar{p}} \int_{t - \bar{\tau}}^t p'(t') dt' \quad (8)$$

Differentiation with respect to t of $\tau - \bar{\tau}$ under the integral sign yields then

$$\frac{d\tau}{dt} = - \frac{n}{\bar{p}} [p'(t) - p'(t - \bar{\tau})] \quad (9)$$

This equation only portrays a part of the combustion chamber phenomenon.

The second part of the derivation of the chamber equation is concerned with the mass balance in the combustion chamber. It begins with the premise that the mass of propellant injected equals the mass burned at an instant τ_t later.

$$\delta \dot{m}_i(t) dt = \delta \dot{m}_b(t + \tau_t) (dt + d\tau) \quad (10)$$

where the subscripts i and b denote injection and burning, respectively.

The time lag τ_t in (10) is approximately equal to and will be later

replaced by its counterpart $\bar{\tau}_t$ in the steady state process. Now let

$$t + \bar{\tau}_t = \tilde{T} \quad \text{or} \quad t = \tilde{T} - \bar{\tau}_t.$$

Eq. (10) becomes

$$\delta \dot{m}_i(\tilde{T} - \bar{\tau}_t) (d\tilde{T} - d\tau) = \delta \dot{m}_b(\tilde{T}) d\tilde{T}$$

or

$$\delta \dot{m}_b(\tilde{T}) = \delta \dot{m}_i(\tilde{T} - \bar{\tau}_t) \left(1 - \frac{d\tau}{d\tilde{T}}\right)$$

Renaming \tilde{T} as t , as the starting instant of the process is immaterial, we have

$$\delta \dot{m}_b(t) = \delta \dot{m}_i(t - \bar{\tau}_t) \left(1 - \frac{d\tau}{dt}\right) \quad (11)$$

Eq. (1) is now modified slightly, with the understanding that in a steady-state process $\delta \dot{m}_i = \delta \dot{m}_b$,

$$\delta \dot{m}_b(t) - \delta \dot{m}_b = \delta \dot{m}_i(t - \bar{\tau}_t) - \delta \dot{m}_i - \frac{d\tau}{dt} \delta \dot{m}_i(t - \bar{\tau}_t) \quad (12)$$

If the injection rate is constant, i.e. $\delta \dot{m}_i(t) = \delta \dot{m}_i = \delta \dot{m}_b$, it follows then

$$\delta \dot{m}_b(t) = \delta \dot{m}_b \left(1 - \frac{d\tau}{dt}\right).$$

The next phase of investigation deals with the dynamics of the combustion chamber. For a non-steady process we can write

$$\dot{m}_p(t) = \dot{m}_e(t) + \frac{d}{dt} M_g(t) \quad (13)$$

where $\dot{m}_b(t)$ is the rate of generation of combustion products, $\dot{m}_e(t)$ is the rate of ejection of gases through the nozzle, and $M_g(t)$ is the mass of gases accumulated inside the chamber. Since $M_g(t)$ is proportional to the chamber pressure, the rate of accumulation of $M_g(t)$ is

$$\frac{d}{dt} M_g(t) = \bar{M}_g \frac{d}{dt} \left[\frac{p(t)}{\bar{p}} \right]$$

where \bar{M}_g is the steady-state value of $M_g(t)$.

Considering the total amount of the combustion product from $t = 0$ to the current instant $t = t$, we have

$$\int_0^t \dot{m}_b(t') dt' = \int_0^{t-\tau_t} \dot{m}_i(t') dt' \quad (14)$$

Differentiation of (14) with respect to t gives

$$\dot{m}_b(t) = \left(1 - \frac{d\tau_t}{dt} \right) \dot{m}_i(t-\tau_t) \quad (15)$$

Substitution of (9) into (15) yields

$$\dot{m}_b(t) = \left\{ 1 + n \left[\frac{p'(t)}{\bar{p}} - \frac{p'(t-\bar{\tau})}{\bar{p}} \right] \right\} \dot{m}_i(t-\tau_t)$$

or, with a slight modification,

$$\frac{\dot{m}_b(t) - \frac{\bar{m} + \bar{\bar{m}}}{\bar{m}}}{\bar{m}} = \left\{ 1 + n \left[\frac{p'(t) - \bar{p} + \bar{p}}{\bar{p}} - \frac{p'(t-\bar{\tau}) - \bar{p} + \bar{p}}{\bar{p}} \right] \right\} \frac{\dot{m}_i(t-\tau_t) - \frac{\bar{m} + \bar{\bar{m}}}{\bar{m}}}{\bar{m}}$$

Denoting

$$\frac{\dot{m}_b(t) - \frac{\bar{m} + \bar{\bar{m}}}{\bar{m}}}{\bar{m}} = \mu_b(t), \quad \frac{\dot{m}_i(t-\tau_t) - \frac{\bar{m} + \bar{\bar{m}}}{\bar{m}}}{\bar{m}} = \mu_i(t-\tau_t) \quad \text{and} \quad \frac{p'(t) - \bar{p}}{\bar{p}} = \phi(t)$$

we have,

$$\mu_b(t) = n [\phi(t) - \phi(t-\bar{\tau})] + \mu_i(t-\tau_t) + n[\phi(t) - \phi(t-\bar{\tau})]\mu_i(t-\tau_t)$$

or, approximately,

$$\mu_b(t) \sim \mu_i(t-\tau_t) + n [\phi(t) - \phi(t-\bar{\tau})] \quad (16)$$

Introducing the gas residence time

$$\theta_g = Mg/\dot{m} , \quad (17)$$

as the time an average element of products of combustion will remain inside the chamber in a steady-state operation before it emerges from the nozzle, Eq. (13) becomes

$$\frac{d\phi}{dz} + \mu(z) = \mu_b(z) \quad (18)$$

$$\text{where } z = t/\theta_g. \quad (19)$$

The ejection rate can be calculated from the steady-state nozzle transfer function [6] as

$$\mu_e(z) \simeq \phi(z) \quad (20)$$

Substitution of (16) into (18) gives the equation of combustion chamber dynamics.

$$\frac{d\phi}{dz} + \phi(z) = \mu_i(z - \bar{\tau}_t) + n [\phi(z) - \phi(z - \bar{\tau})] \quad (21)$$

where τ_t has been approximated by the steady-state value $\bar{\tau}_t$ and has also been non-dimensionalized by the use of θ_g .

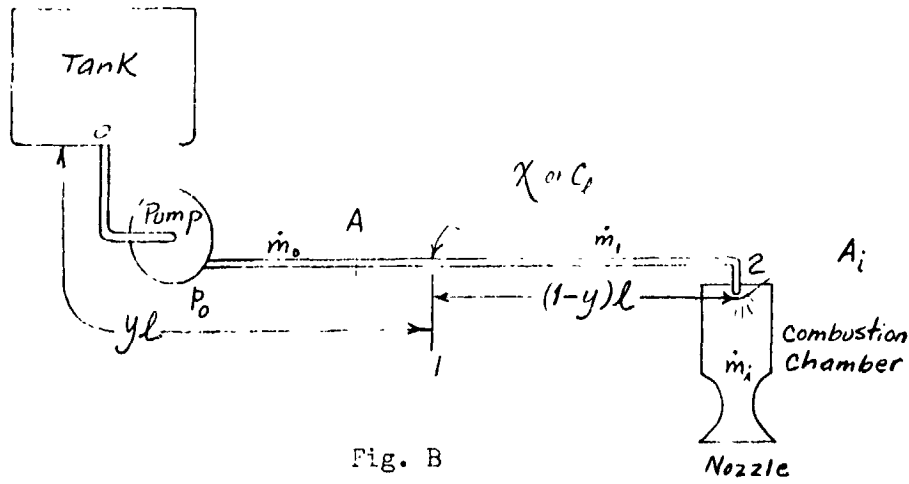
The term $\mu_i(z - \bar{\tau}_t)$ in (21) can not be determined without an examination of the mechanics of the feed system of the rocket engine.

Obviously, if the injection rate is independent of the pressure oscillations in the chamber, (21) reduces to its simplest form.

$$\frac{d\phi}{dz} + (1-n) \phi(z) = -n \phi(z - \bar{\tau}) \quad (21)$$

Derivation of Feed System Equation

A schematic feed system for a monopropellant rocket engine as shown as in Fig. 2^b can be mathematically represented by several simple component equations, each of which portrays a specific portion of the operation.



$$\text{Pump:} \quad \frac{p_0 - \bar{p}_0}{\bar{p}_0} = -D \frac{\dot{m}_0 - \bar{\dot{m}}_0}{\bar{\dot{m}}_0} \quad (22)$$

$$\text{Line 0-1:} \quad \dot{m}_0 - \dot{m}_1 = \rho_0 \times \frac{dp_1}{dt} \quad (23)$$

$$p_0 - p_1 = \frac{\gamma l}{A} \frac{d\dot{m}_0}{dt} \quad (24)$$

$$\text{Line 1-2:} \quad p_1 - p_2 = \frac{(1-\gamma)l}{A} \frac{d\dot{m}_1}{dt} \quad (25)$$

$$\text{Injector plate:} \quad p_2 - p_1 = \frac{1}{2} \frac{\dot{m}_i^2}{\rho_0 A_i^2} \quad (26)$$

$$\dot{m}_1 = \dot{m}_i \quad (\text{No feedback control}) \quad (27)$$

where D and X are the proportional constant of the pump and the equivalent spring constant of the feed line, respectively. X is closely related to the feed line capacitance C_L .

H.S. Tsien [3] obtained a differential equation from (22) - (27) which relates the feed system dynamics with the combustion chamber phenomenon.

$$\begin{aligned} & P\left[\phi + DE\left(P + \frac{1}{2}\right)\frac{d\phi}{dz} + JEy\frac{d^2\phi}{dz^2}\right] \\ & + \left[1 + D\left(P + \frac{1}{2}\right)\right]\mu_i + \left[DE\left(P + \frac{1}{2}\right) + J\right]\frac{d\mu_i}{dz} \\ & + [DJE(1-y)\left(P + \frac{1}{2}\right) + JEy]\frac{d^2\mu_i}{dz^2} + J^2Ey(1-y)\frac{d^3\mu_i}{dz^3} = 0 \end{aligned} \quad (28)$$

where

$$P = \frac{1}{2} \frac{\bar{p}}{\Delta\bar{p}} \quad (\text{pressure drop parameter})$$

$$E = \frac{2\Delta\bar{p}_0 X}{\bar{m} \theta_g} \quad (\text{Elasticity parameter})$$

$$J = \frac{\bar{L}_m}{2\Delta\bar{p}A\theta_g} \quad (\text{Inertia parameter})$$

Simplification of (28) is possible for specific cases:

I. Constant feed pressure, $D = 0$

$$P\left(\phi + JEy\frac{d^2\phi}{dz^2}\right) + \mu_i + J\frac{d\mu_i}{dz} + JEy\frac{d^2\mu_i}{dz^2} + J^2Ey(1-y)\frac{d^3\mu_i}{dz^3} = 0 \quad (29)_I$$

II. Line elasticity or line capacitance concentrated at injector plate, $y=1$

$$P\left(\phi + JE\frac{d^2\phi}{dz^2}\right) + \mu_i + J\frac{d\mu_i}{dz} + JE\frac{d^2\mu_i}{dz^2} = 0 \quad (29)_{II}$$

III. Constant feed rate, $D \rightarrow \infty$

$$PE \frac{d\phi}{dz} + \mu_i + E \frac{d\mu_i}{dz} + (1-y) JE \frac{d^2\mu_i}{dz^2} = 0 \quad (29)_{III}$$

IV. Line elasticity or Line capacitance concentrated at the tank end of line, $y = 0$.

$$P \left[\phi + DE \left(P + \frac{1}{2} \right) \frac{d\phi}{dz} \right] + \left[1 + D \left(P + \frac{1}{2} \right) \right] \mu_i + \left[DE \left(P + \frac{1}{2} \right) + J \right] \frac{d\mu_i}{dz} + DJE \left(P + \frac{1}{2} \right) \frac{d^2\mu_i}{dz^2} = 0 \quad (29)_{IV}$$

For each special case (four sample cases are shown above) the feed system equation $F(\mu_i, \phi) = 0$ (29) must be solved simultaneously with the chamber equation $C(\phi, \mu_i) = 0$ (21). If one is interested in the stability aspects of the problem, only the characteristic equation is of importance. Some sample analyses leading to this chamber equation will be given below.

Case I. Define a differential operator

$$\mathcal{D} = 1 + J \frac{d}{dz} + JEy \frac{d^2}{dz^2} + J^2Ey (1-y) \frac{d^3}{dz^3} \quad (30)$$

Then $(29)_I$ becomes for the instant of $z - \bar{\tau}_t$

$$P \left[1 + JEy \frac{d^2}{dz^2} \right] \phi_{(z-\bar{\tau})} + \mathcal{D} \mu_i(z - \bar{\tau}_t) = 0 \quad (31)$$

Applying the differential operator \mathcal{D} defined by (30) into (21) we have

$$\mathcal{D} \left[(1-n)\phi(z) - n\phi(z-\bar{\tau}) + \frac{d\phi}{dz} \right] = \mathcal{D} \mu_i(z - \bar{\tau}_t) \quad (32)$$

Substitution of (32) into (31) yields a differential equation in ϕ only.

$$\begin{aligned} & \left[1 + J \frac{c}{dz^2} + JEy \frac{d^2}{dz^2} + J^2Ey(1-y) \frac{d^3}{dz^3} \right] [(1-n)\phi(z) + \frac{d\phi}{dz} - n\phi(z-\bar{\tau})] \\ & + P \left[1 + JEy \frac{d^2}{dz^2} \right] \phi(z-\bar{\tau}) = 0 \end{aligned}$$

or

$$\begin{aligned} & \left\{ (1-n) + [1 + (1-n)J] \frac{d}{dz} + J [1 + (1-n)Ey] \frac{d^2}{dz^2} + JEy [(1-n)J(1-y) + 1] \frac{d^3}{dz^3} \right. \\ & \left. + J^2Ey (1-y) \frac{d^4}{dz^4} \right\} \phi(z) \\ & + [P - n - nJ \frac{d}{dz} + (P-n)JEy \frac{d^2}{dz^2} - nJ^2Ey(1-y) \frac{d^3}{dz^3}] \phi(z-\bar{\tau}) = 0 \quad (33) \end{aligned}$$

Equation (33) can be written symbolically

$$L_1 [\phi(z)] + L_2 [\phi(z-\bar{\tau})] = 0 \quad (34)$$

where

$$\begin{aligned} L_1 &= (1-n) + [1 + (1-n)J] \frac{d}{dz} + J [1 + (1-n)Ey] \frac{d^2}{dz^2} \\ &+ JEy [(1-n)J(1-y) + 1] \frac{d^3}{dz^3} + J^2Ey (1-y) \frac{d^4}{dz^4}, \quad (35) \end{aligned}$$

and

$$L_2 = (P-n) - nJ \frac{d}{dz} + (P-n)JEy \frac{d^2}{dz^2} - nJ^2Ey (1-y) \frac{d^3}{dz^3}.$$

Now we relate the function with a retarded variable $(z-\bar{\tau})$ with $\phi(z)$ by the use of Laplace transform, namely,

$$\phi(s) = \int_0^{\infty} e^{-sz} \phi(z) dz$$

and

$$\int_0^{\infty} e^{-sz} \phi(z-\bar{t}) dz = e^{-s\bar{t}} \phi(s) \quad (36)$$

Thus,

$$\begin{aligned} & \left\{ (1-n) + [1 + (1-n)J]s + J [1 + (1-n)Ey]s^2 + JEy [1 + (1-n)J(1-y)]s^3 \right. \\ & \quad \left. + J^2Ey(1-y)s^4 \right\} \phi(s) - [1 + (1-n)J] \phi(0) \\ & \quad - [1 + (1-n)Ey] \left[s\phi(0) + \frac{d}{dz}\phi \Big|_{z=0} \right] \\ & \quad - JEy [1 + (1-n)J(1-y)] \left[s^2\phi(0) + s \frac{d}{dz}\phi \Big|_{z=0} + \frac{d^2}{dz^2}\phi \Big|_{z=0} \right] \\ & \quad - J^2Ey(1-y) \left[s^3\phi(0) + s^2 \frac{d}{dz}\phi \Big|_{z=0} + s \frac{d^2}{dz^2}\phi \Big|_{z=0} + \frac{d^3}{dz^3}\phi \Big|_{z=0} \right] \\ & \quad + e^{-s\bar{t}} \left\{ [(P-n) - nJs + (P-n)JEy s^2 - nJ^2Ey(1-y)s^3] \phi(s) \right. \\ & \quad \left. + nJ\phi(0) - (P-n)JEy \left[s\phi(0) + \frac{d}{dz}\phi \Big|_{z=0} \right] \right. \\ & \quad \left. + nJ^2Ey(1-y) \left[s^2\phi(0) + s \frac{d}{dz}\phi \Big|_{z=0} + \frac{d^2}{dz^2}\phi \Big|_{z=0} \right] \right\} = 0 \quad (37) \end{aligned}$$

Solving for $\phi(s)$, we have

$$\Phi(s) = \phi_1(s)/\phi_2(s) \quad (38)$$

where

$$\begin{aligned} \phi_1(s) &= [1 + (1-n)J] \phi(0) + [1 + (1-n)Ey] [s\phi(0) + \phi'(0)] + JEy [1 + (1-n)J(1-y)] \\ & \quad \cdot [s^2\phi(0) + s\phi'(0) + \phi''(0)] + J^2Ey(1-y) [s^3\phi(0) + s^2\phi'(0) + s\phi''(0) + \phi'''(0)] \\ & \quad - e^{-s\bar{t}} \left\{ nJ\phi(0) - (P-n)JEy [s\phi(0) + \phi'(0)] + nJ^2Ey(1-y) [s^2\phi(0) + s\phi'(0) + \phi''(0)] \right\} \end{aligned}$$

$$\begin{aligned}\phi_2(s) = & (1-n) + [1+(1-n)J] s + J[1+(1-n)Ey] s^2 \\ & + JEy[1+(1-n)J(1-y)] s^3 + J^2Ey(1-y)s^4 \\ & - e^{-s\bar{\tau}} [(P-n) - nJs + (P-n)JEys^2 - nJ^2Ey(1-y)s^3]\end{aligned}$$

Eq (38) can now be inverted. The inversion process involves the evaluation of the residues at the poles inside a contour which encloses all the poles of the integrand. Since there are no poles other than those introduced by the vanishing of the denominator, we only need to set it to zero and seek its roots, either real or complex. Hence

$$\begin{aligned}(1-n) + [1+(1-n)J] s + J[1+(1-n)Ey] s^2 + JEy[1+(1-n)J(1-y)] s^3 + J^2Ey(1-y)s^4 \\ = e^{-s\bar{\tau}} [(P-n) - nJs + (P-n)JEys^2 - nJ^2Ey(1-y)s^3]\end{aligned}\quad (39)$$

If $y = 1$, such that the line capacitance is concentrated at the injector plate end of the line, (39) further reduces to

$$\frac{(1-n) + [1 + (1-n)J] s + J[1 + (1-n)E]s^2 + JEs^3}{P - n - nJs + (P-n) JE s^2} = e^{-\bar{\tau}s}\quad (40)$$

For given values of J, E, P, N and $\bar{\tau}$, the roots (either real or complex) can be solved from (40). Since these roots will give rise in the solution $\phi(z)$ to such terms as

$$c_i e^{s_i z} \quad i = 0, 1, 2, \dots \infty \quad (41)$$

We are certain that a real positive root S_i or a positive real part of a complex root will cause $\phi(z)$ to increase without bound as z increases. In other words, the existence of such a root signifies instability, whereas the existence of a negative real root S_i or a negative real part of a complex root indicates stability.

Case III. Constant feed rate $D \rightarrow \infty$

$$\text{Define } \mathcal{P} = 1 + E \frac{d}{dz} + (1-y) JE \frac{d^2}{dz^2} \quad (42)$$

then (29)III becomes for the instant of $z = \bar{\tau}_t$

$$PE \frac{d}{dz} \phi(z - \bar{\tau}) + \mathcal{P}\mu_i(z - \bar{\tau}_t) = 0 \quad (43)$$

But $\mathcal{P}\mu_i(z - \bar{\tau}_t)$ is, from (21),

$$\mathcal{P}\mu_i(z - \bar{\tau}_t) = \mathcal{P}[(1 - n + \frac{d}{dz})\phi(z) + n\phi(z - \bar{\tau})] \quad (44)$$

Hence

$$PE \frac{d}{dz} \phi(z - \bar{\tau}) + [1 + E \frac{d}{dz} + (1 - y) JE \frac{d^2}{dz^2}][(1 - n + \frac{d}{dz})\phi(z) + n\phi(z - \bar{\tau})] = 0$$

Simplifying,

$$\begin{aligned} & (1 - n + \frac{d}{dz})[1 + E \frac{d}{dz} + (1 - y) JE \frac{d^2}{dz^2}]\phi(z) \\ & + \{PE \frac{d}{dz} + n[1 + E \frac{d}{dz} + (1 - y) JE \frac{d^2}{dz^2}]\}\phi(z - \bar{\tau}) = 0 \end{aligned} \quad (45)$$

The counterpart of Eq. (39) for this case is

$$(1 + Es + (1 - y) JEs^2)[1 + s - n + n e^{-\bar{\tau}s}] + PE s e^{-\bar{\tau}s} = 0 \quad (46)$$

Eq. (46) can be solved for s for s real or for s , complex. For real s , we

transpose(46) as

$$e^{-\bar{\tau}s} = - \frac{(1 - n + s)[1 + Es + (1 - y)JEs^2]}{PEs + n[1 + Es + (1 - y)JEs^2]} \quad (47)$$

Graphical procedure must be used to construct curves for the left and right sides of (47) for various values of the parameters n, E, y, J , and P . The intersections of these curves locate the roots. Again, positive roots indicate stability, otherwise, instability.

For s complex, we substitute in (46)

$$s = \alpha + i\omega$$

resulting in

$$e^{-\alpha\bar{\tau}}(\cos \bar{\tau}\omega - i \sin \bar{\tau}\omega) = - \frac{[(1 - n + \alpha)A_1 - \omega A_2] + i[\omega A_1 + (1 - n + \alpha)A_2]}{(PE\alpha + nA_1) + i(PE\omega + nA_2)} \quad (48)$$

where $A_1 = 1 + E\alpha + (1 - y)JE(\alpha^2 - \omega^2)$,

$$A_2 = E\omega + 2(1 - y)JE\alpha\omega. \quad (49)$$

Equate the real and imaginary parts in (48), we obtain

$$e^{-\alpha\bar{\tau}} \cos \bar{\tau}\omega = \frac{(PE\alpha + nA_1)[(1-n+\alpha)A_1 - \omega A_2] + (PE\omega + nA_2)[\omega A_1 + (1-n+\alpha)A_2]}{(PE\alpha + nA_1)^2 + (PE\omega + nA_2)^2} \quad (50)$$

$$e^{-\alpha\bar{\tau}} \sin \bar{\tau}\omega = \frac{(PE\alpha + nA_1)[(1-n+\alpha)A_2 + \omega A_1] - (PE\omega + nA_2)[- \omega A_2 + (1-n+\alpha)A_1]}{(PE\alpha + nA_1)^2 + (PE\omega + nA_2)^2}$$

Further simplification yields

$$-\tan \tau\omega = \frac{(PE\alpha + nA_1) \omega A_1 + (1-n+\alpha)A_2 - (PE\omega + nA_2) (1-n+\alpha)A_1 - \omega A_2}{-(PE\alpha + nA_1) \omega A_2 - (1-n+\alpha)A_1 + (PE\omega + nA_2) (1-n+\alpha)A_2 - \omega A_1} \quad (51)$$

$$e^{-2\alpha\tau} = \frac{[(1-n+\alpha)^2 + \omega^2][A_1^2 + A_2^2]}{(PE\alpha + nA_1)^2 + (PE\omega + nA_2)^2} \quad (52)$$

The case of neutral stability is characterized by $\alpha = 0$. Setting $\alpha = 0$ in (51) and (52) we obtain

$$-\tan \tau\omega = \frac{nA_1^0[\omega A_1^0 + (1-n)A_2^0] - (PE\omega + nA_2^0)[(1-n)A_1^0 - \omega A_2^0]}{-nA_1^0[\omega A_2^0 - (1-n)A_1^0] + (PE\omega + nA_2^0)[(1-n)A_2^0 - \omega A_1^0]} \quad (53)$$

$$1 = \frac{[(1-n)^2 + \omega^2][(A_1^0)^2 + (A_2^0)^2]}{(nA_1^0)^2 + (PE\omega + nA_2^0)^2} \quad (54)$$

where $A_1^0 = 1 - J'E\omega^2$

$$A_2^0 = E\omega$$

$$J' = (1 - \gamma)J.$$

Simplifying, (53), (54) becomes

$$-\tan \tau\omega = \frac{\omega\psi - (1-n) + (P+n) \left[\frac{\omega}{n\psi} + \frac{1-n}{n} \right]}{\omega + (1-n)\psi + (P+n) \left[\frac{\omega}{n} + \frac{1-n}{n\psi} \right]} \quad (55)$$

and

$$(P+n)^2 - n^2 = [\omega^2 + (1-n)^2 - n^2](\psi^2 + 1) \quad (56)$$

where

$$\psi = J\omega - \frac{1}{E\omega}.$$

THE PRESENT MODEL

The low frequency combustion stability model described here is an extension of Crocco and Cheng's (4) analysis of the monopropellant system to the bipropellant case. The model considers the injector capacitance and inertance as well as the resistance of both propellant systems and allows for separate and distinct time lags for each propellant. Previous bi-propellant models do not consider the effects of injector capacitance and inertance. The time lags are the time intervals between the fuel and oxidizer injection and the assumed sudden conversion to exhaust products; they include all the physical and chemical processes in the conversion such as heating, vaporization, mixing, and reaction.

ANALYSIS

The monopropellant, single time lag model of Crocco and Cheng is modified to accommodate the bipropellant case by adding a term accounting for the second propellant to the equation governing the dynamics of the gas flow in the combustion chamber and adding a new equation representing the dynamics of the second feed system. The modified equation for the chamber dynamics in dimensionless form is

$$\begin{aligned} \frac{d}{dt} \delta p_c(t) + \delta p_c(t) = & \delta \dot{W}_o (t - \tau_o) + \delta \dot{W}_f (t - \tau_f) \\ & + n [\delta p_c(t) - \delta p_c(t-\tau)] \end{aligned} \quad (57)$$

assuming the the pressure and temperature at any given instant are constant throughout the combustion chamber and the time lag is constant for all propellant elements. The dimensionless chamber pressure, F_c and flow rates \dot{W}_o and \dot{W}_f are

defined in terms of their steady-state values and the dimensionless time, t , time lags for the oxidizer and the fuel, τ_o and τ_f , and sensitive time lag, $\bar{\tau}$, are defined in terms of the gas residence time, θ_g . (Symbols are listed at the front of the report and θ_g is defined in Appendix A.)

Two dimensionless equations representing the dynamics of the feed system, one for the oxidizer and one for the fuel assuming constant feed pressure at the injector inlet and all capacitance and inertance located at the injector are

$$P_o(1+J_o E_o D^2) \delta p_c(t) + (1+J_o D + J_o E_o D^2) \delta \dot{W}_o(t) = 0 \quad (58)$$

$$P_f(1+J_f E_f D^2) \delta p_c(t) + (1+J_f D + J_f E_f D^2) \delta \dot{W}_f(t) = 0 \quad (59)$$

where D is a differential operator, and the dimensionless inertance, J_o and J_f , capacitance E_o and E_f and pressure drop parameters, P_o and P_f , are defined in Appendix A. Substituting equations 2 and 3 into equation 1 and applying Laplace transformation to the result yields

$$\begin{aligned} & [S+1-n+ne^{-\tau S}][1+J_o S+J_o E_o S^2][1+J_f S+J_f E_f S^2] \\ & = -e^{-\tau_o S} P_o [1+J_o E_o S^2] [1+J_f S+J_f E_f S^2] - e^{-\tau_f S} P_f [1+J_f E_f S^2] [1+J_o S+J_o E_o S^2] \end{aligned} \quad (60)$$

Substituting $\alpha+i\omega$ for S in equation (60) and equating the real and imaginary parts of both sides results in two simultaneous equations,

$$\begin{aligned} (GH-KN)(\alpha+1)-(KH+GN)\omega &= -P_o e^{-\alpha\tau_o} \left\{ \cos \omega\tau_o [MH-N(2\alpha\omega J_o E_o)] + \sin \omega\tau_o [MH+H(2\alpha\omega J_o E_o)] \right\} \\ & - P_f e^{-\alpha\tau_f} \left\{ \cos \omega\tau_f [RG-K(2\alpha\omega J_f E_f)] + \sin \omega\tau_f [KR+G(2\alpha\omega J_f E_f)] \right\} \end{aligned} \quad (61)$$

$$\begin{aligned} (GH-KN)\omega + (KH+GN)(\alpha+1) &= -P_o e^{-\alpha\tau_o} \left\{ \cos \omega\tau_o [MN+H(2\alpha\omega J_o E_o)] \right. \\ & \left. - \sin \omega\tau_o [MH-N(2\alpha\omega J_o E_o)] \right\} \\ & - P_f e^{-\alpha\tau_f} \left\{ \cos \omega\tau_f [KR+G(2\alpha\omega J_f E_f)] - \sin \omega\tau_f [RG-K(2\alpha\omega J_f E_f)] \right\} \end{aligned} \quad (62)$$

$$\text{where: } G = 1 + J_o \alpha + J_o E_o (\alpha^2 - \omega^2)$$

$$H = 1 + J_f \alpha + J_f E_f (\alpha^2 - \omega^2)$$

$$K = J_o \omega + 2J_o E_o \alpha \omega$$

$$N = J_f \omega + 2J_f E_f \alpha \omega$$

$$M = 1 + J_o E_o (\alpha^2 - \omega^2)$$

$$R = 1 + J_f E_f (\alpha^2 - \omega^2)$$

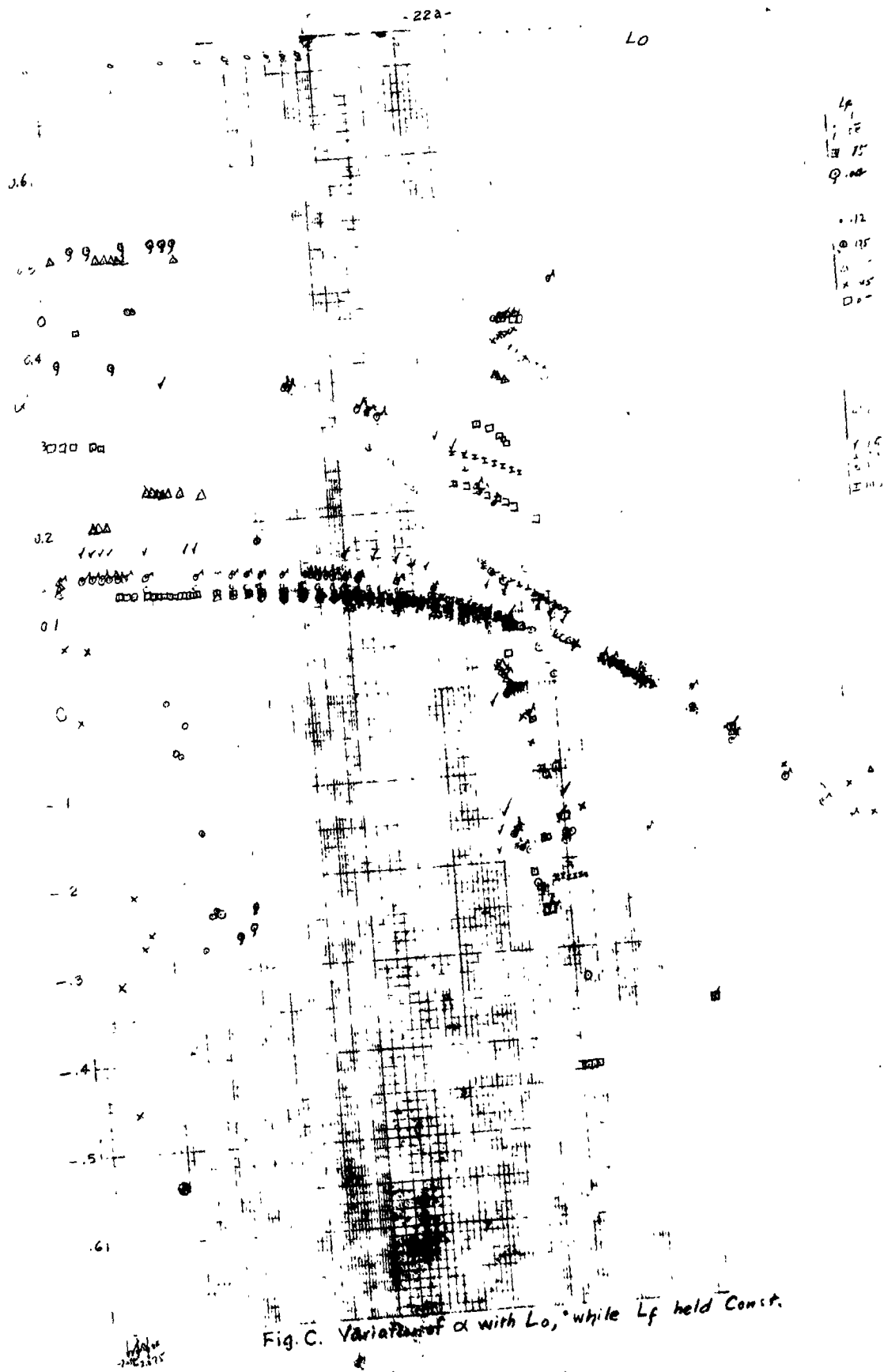
For reasons to be stated later, the interaction index n and the vaporization time $\bar{\tau}$ have been set to zero in Equations (61) and (62).

Equations (61) and (62) can be solved simultaneously† for α and ω to evaluate the stability of any combustor design once the pressure drop, inertance, capacitance and time lags are known*. The magnitude of α indicates incidentally, the proneness of the system to instability (if $\alpha > 0$) or to stability (if $\alpha < 0$).

The time lags, τ_o and τ_f are by Crocco's definition the total time lags and are composed of a constant, steady-state (insensitive) portion and a variable (sensitive) portion $\bar{\tau}$. Rigorous analyses would take into account this time-dependent or sensitive portion of the time lag; however, for most applications it can be neglected because it is small compared to the total time lag. Once the sensitive time lag is taken as zero it follows that the interaction index must also be zero because zero sensitive time lag requires that the burning rate be independent of chamber pressure.

† The programs written for this purpose are listed in Appendix C. The calculations which underlie the solutions for α and ω are discussed in Appendix B.

* The curves showing variations of α with L_o (V_o) while L_f (V_f) is held constant are shown in Figures C (D).



-22b-

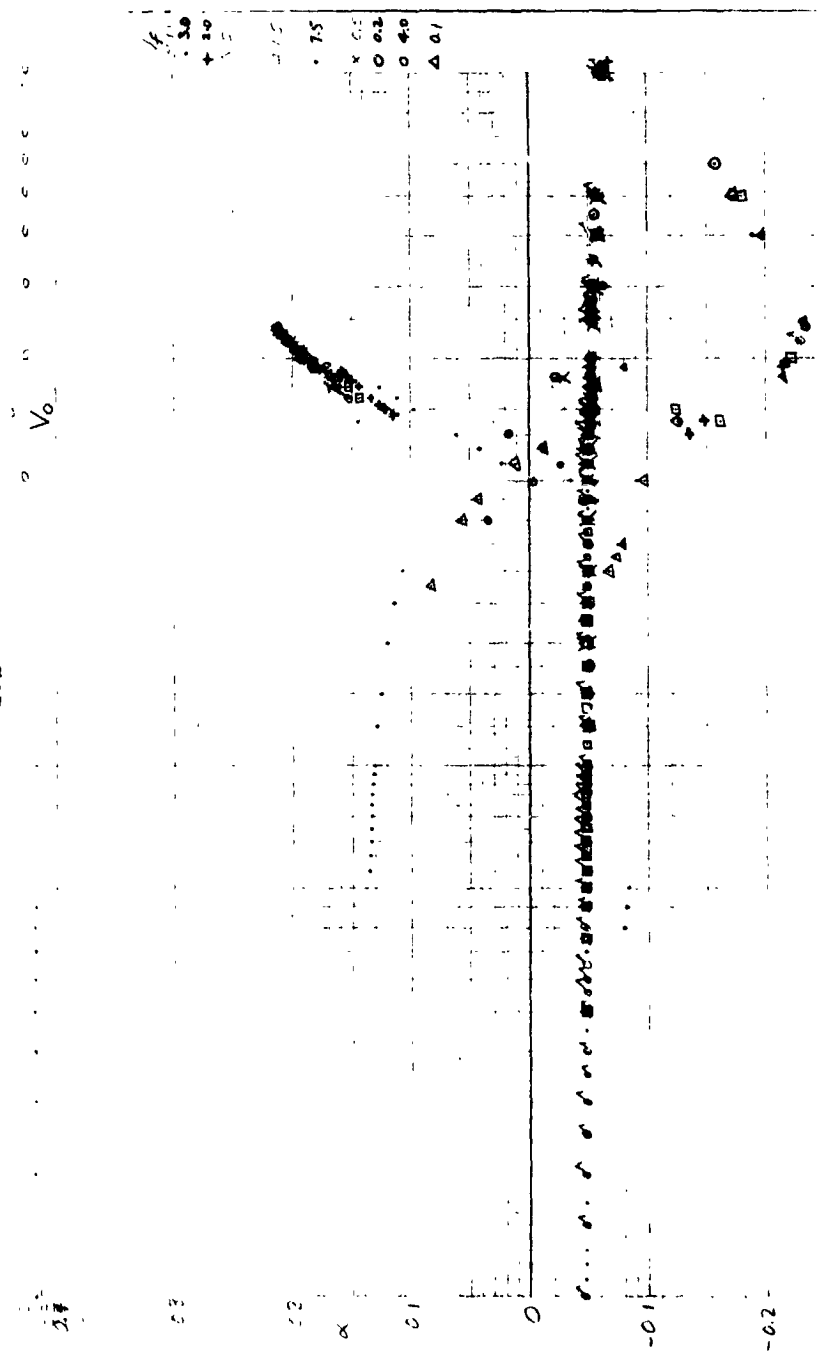


Fig D. Variation of α with V_0 , while V_f held Const.

The total time lag is defined as the time increment between injection of a propellant and its conversion into combustion products. Certainly this time lag is not the same for all propellant elements. It is therefore customary to define an average time lag for each propellant which may be done by determining the lapsed travel time between injection and the axial position where combustion is assumed to take place. The methods of Priem [7] can be used to determine the position of the combustion front.

DISCUSSIONS AND RESULTS

In order to illustrate the effects of capacitance, inertance and resistance of the systems on stability, equations 5 and 6 were solved for the neutral stability boundaries ($\alpha = 0$). Figures 1 through 3 show the neutral stability boundaries in terms of the fuel and oxidizer orifice lengths for three different oxidizer pressure drops. Fuel and oxidizer cavity volume, fuel pressure drop and time lags are held constant. Figures 4 and 5 show the neutral stability boundaries for two different fuel pressure drops while oxidizer pressure drop is held constant along with the cavity volumes and time lags. Because more than one pair of roots satisfy equations 5 and 6 multiple stable and unstable zones exist. The fuel and oxidizer pressure drops affect these zones but unfortunately no trends are apparent.

Figures 6 through 8 show the neutral stability boundaries in terms of the fuel and oxidizer cavity volumes for three different oxidizer pressure drops. Fuel and oxidizer orifice lengths, fuel pressure drop and time lags are held constant. Figures 9 and 10 show the neutral stability boundaries for two different fuel pressure drops while oxidizer pressure drop is held constant along with the orifice lengths and time lags. Although multiple stable and unstable zones exist as in figures 1 through 5, it appears that intermediate values of oxidizer and fuel pressure drops (figures 7 and 9, respectively) result in the narrowest unstable zones. Thus, if an operating point were selected which is in an unstable zone of figure 6, the system could be stabilized by reducing either the oxidizer pressure drop to 124 psi, or the fuel pressure drop to 87 psi (figures 7 and 9). Further reduction in either oxidizer or fuel pressure drop results in the return to unstable operation. This result cannot be predicted by either single time lag models or double time lag models which do not include injector inertance and capacitance.

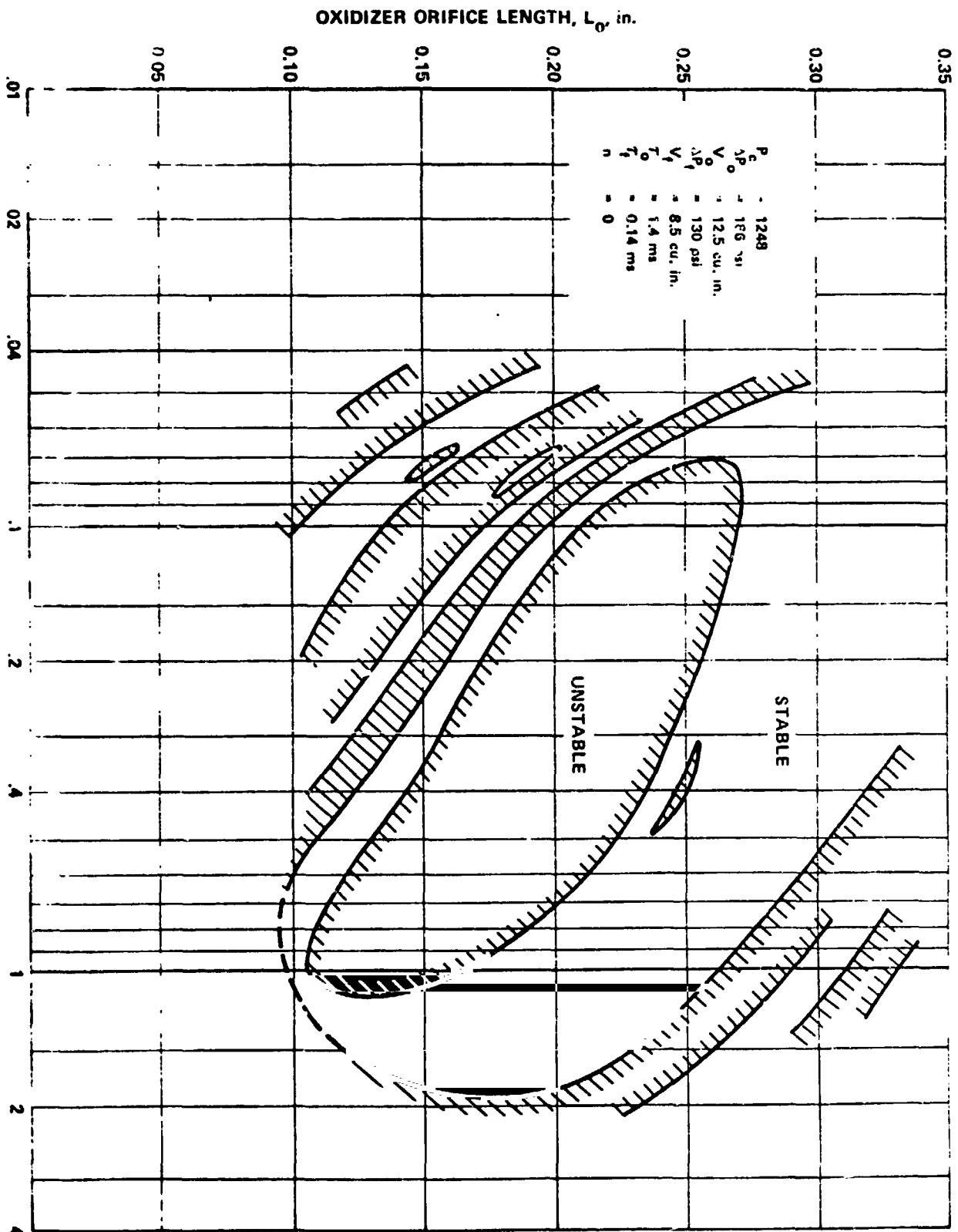


Fig. 1. FUEL ORIFICE LENGTH, L_f , in.

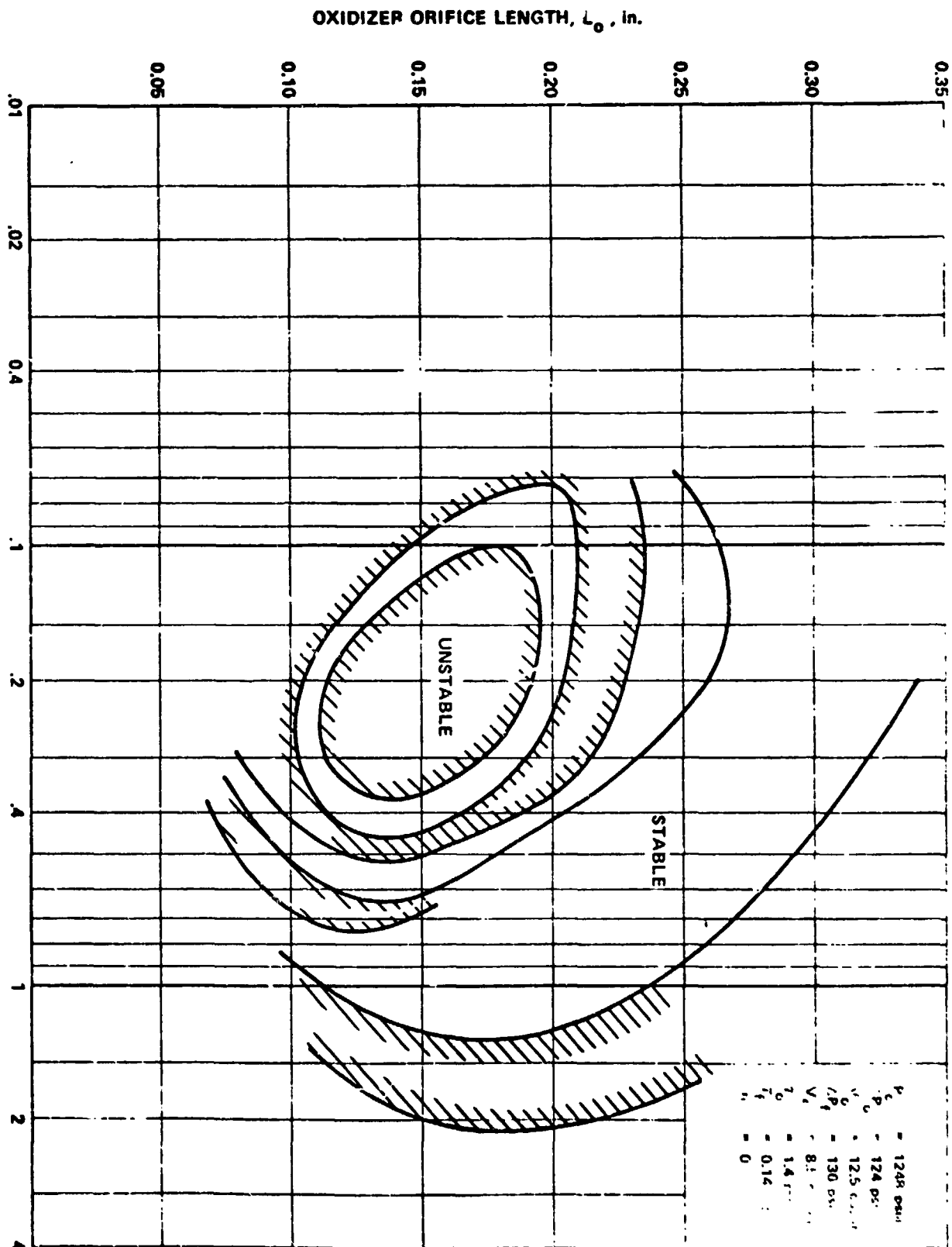
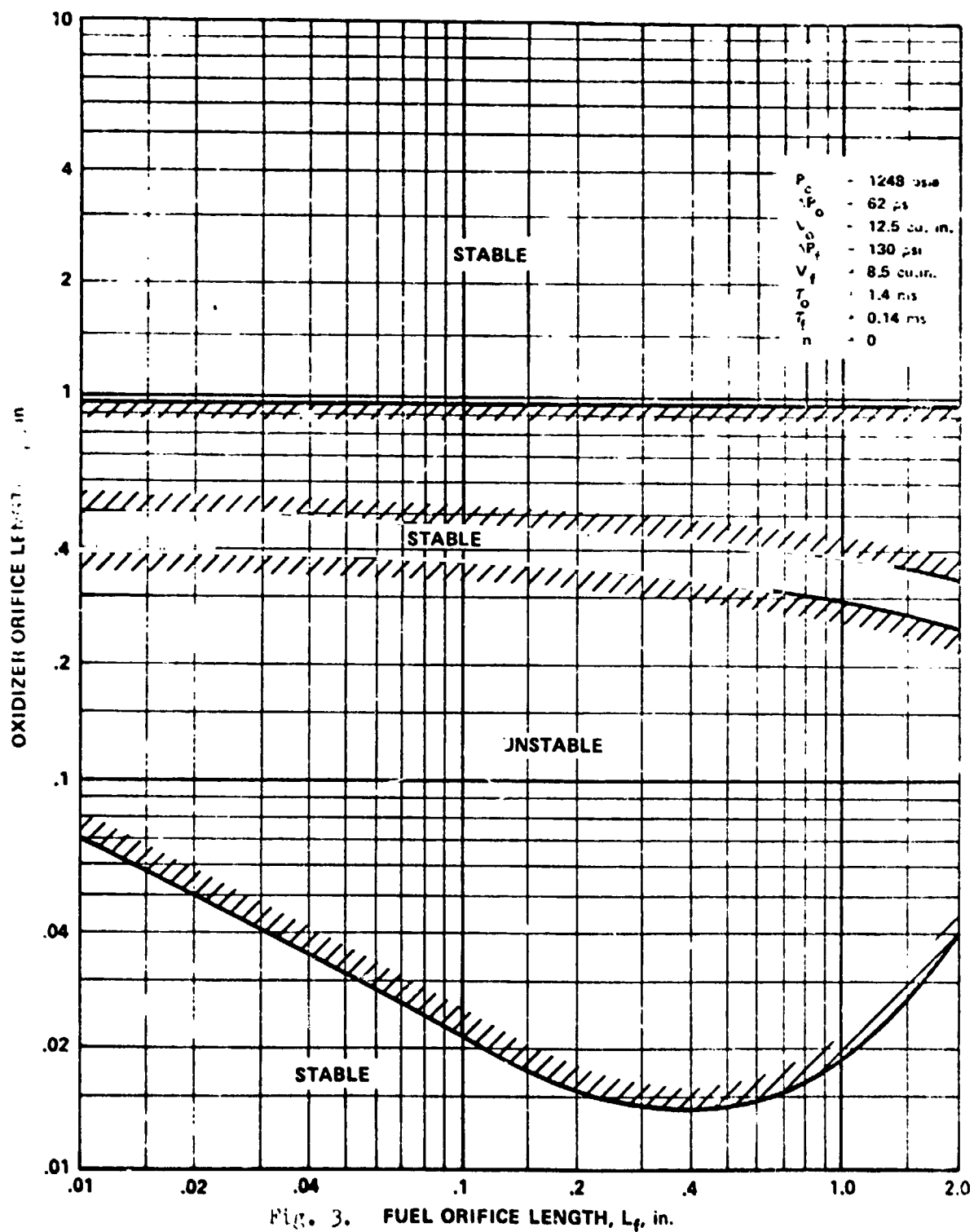


FIG. 2. FUEL ORIFICE LENGTH, L_f , in.



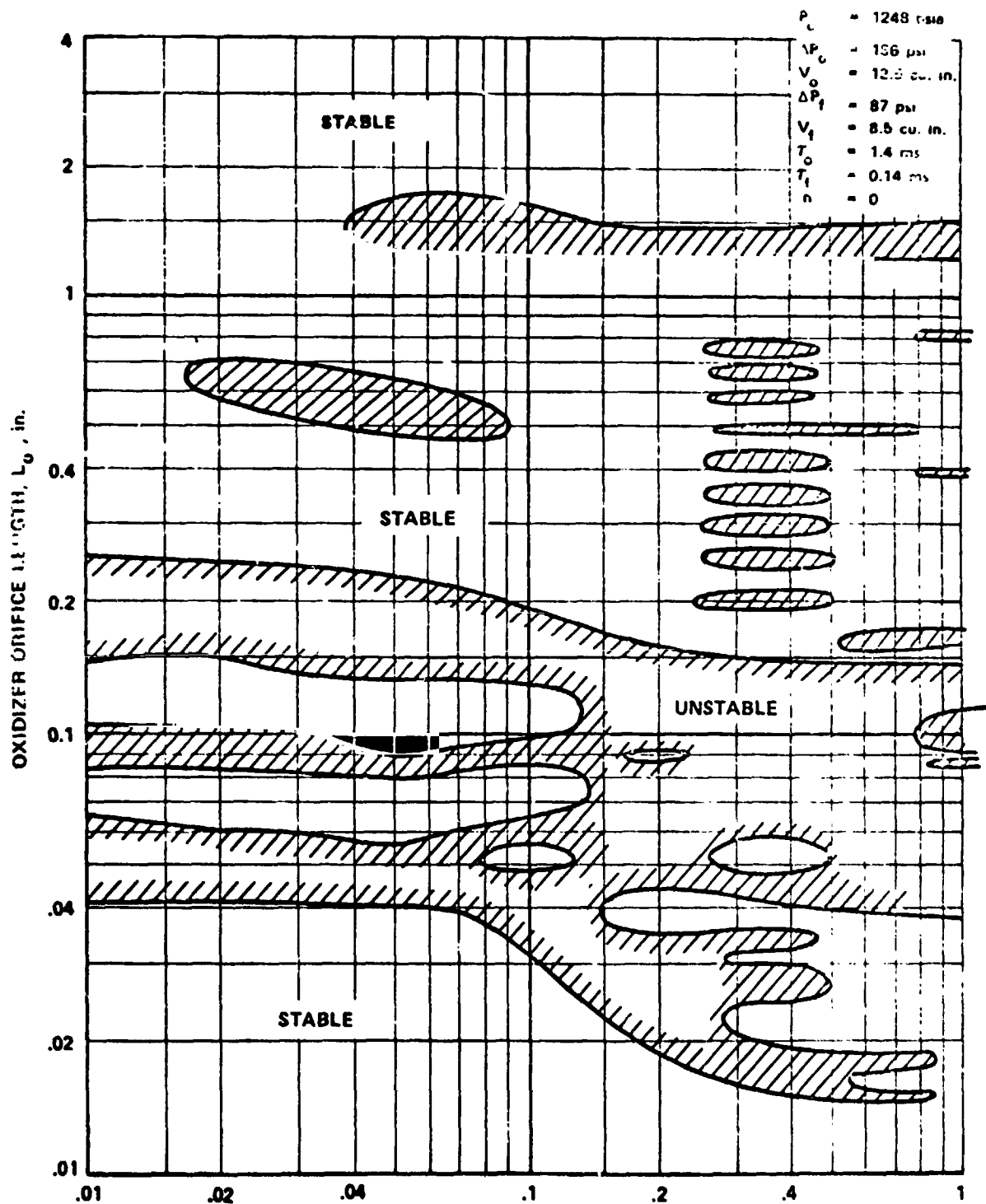


Fig. 4. FUEL ORIFICE LENGTH, L_f , in.

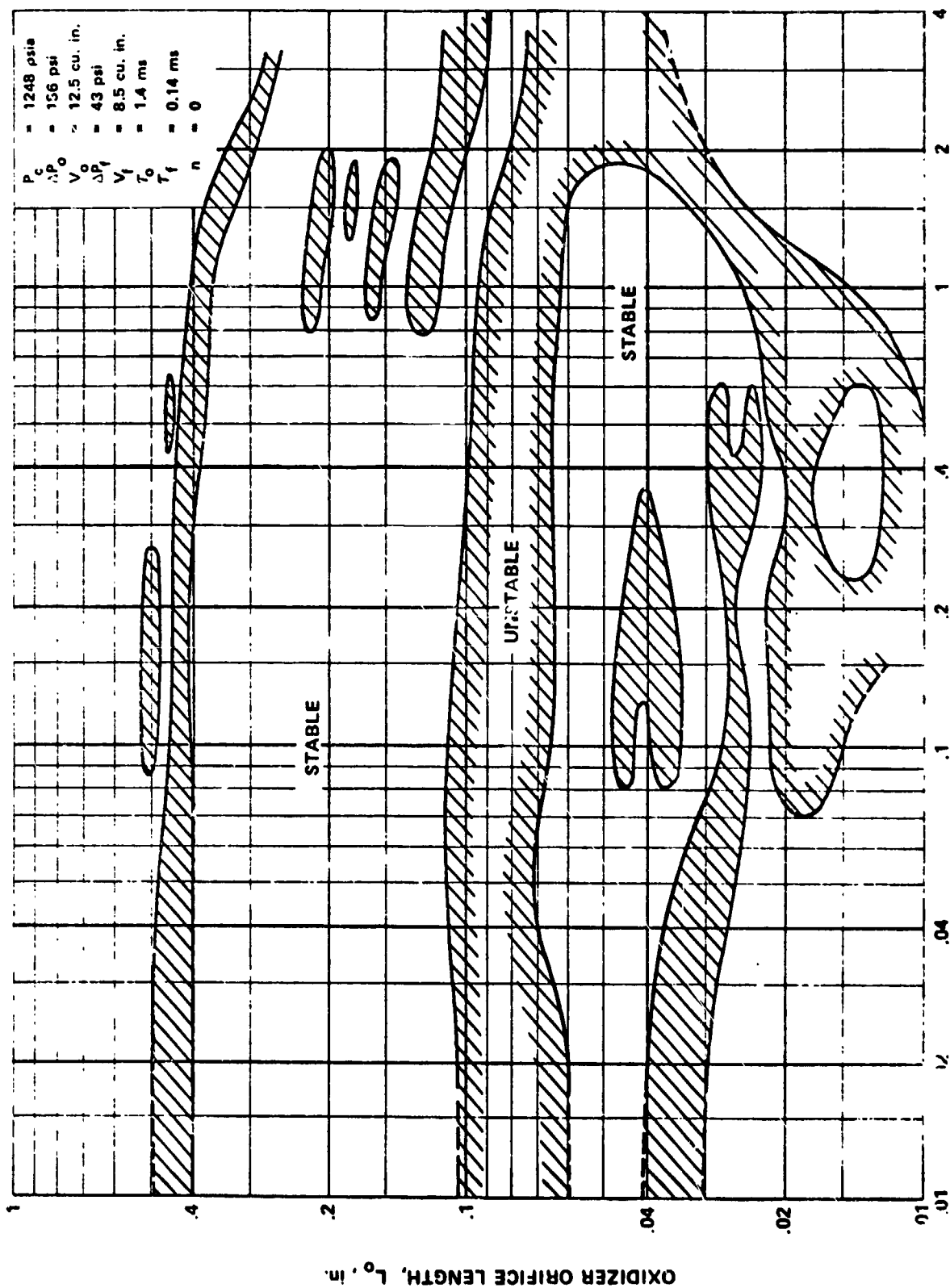


Fig. 5. FUEL ORIFICE LENGTH, L_f , in.

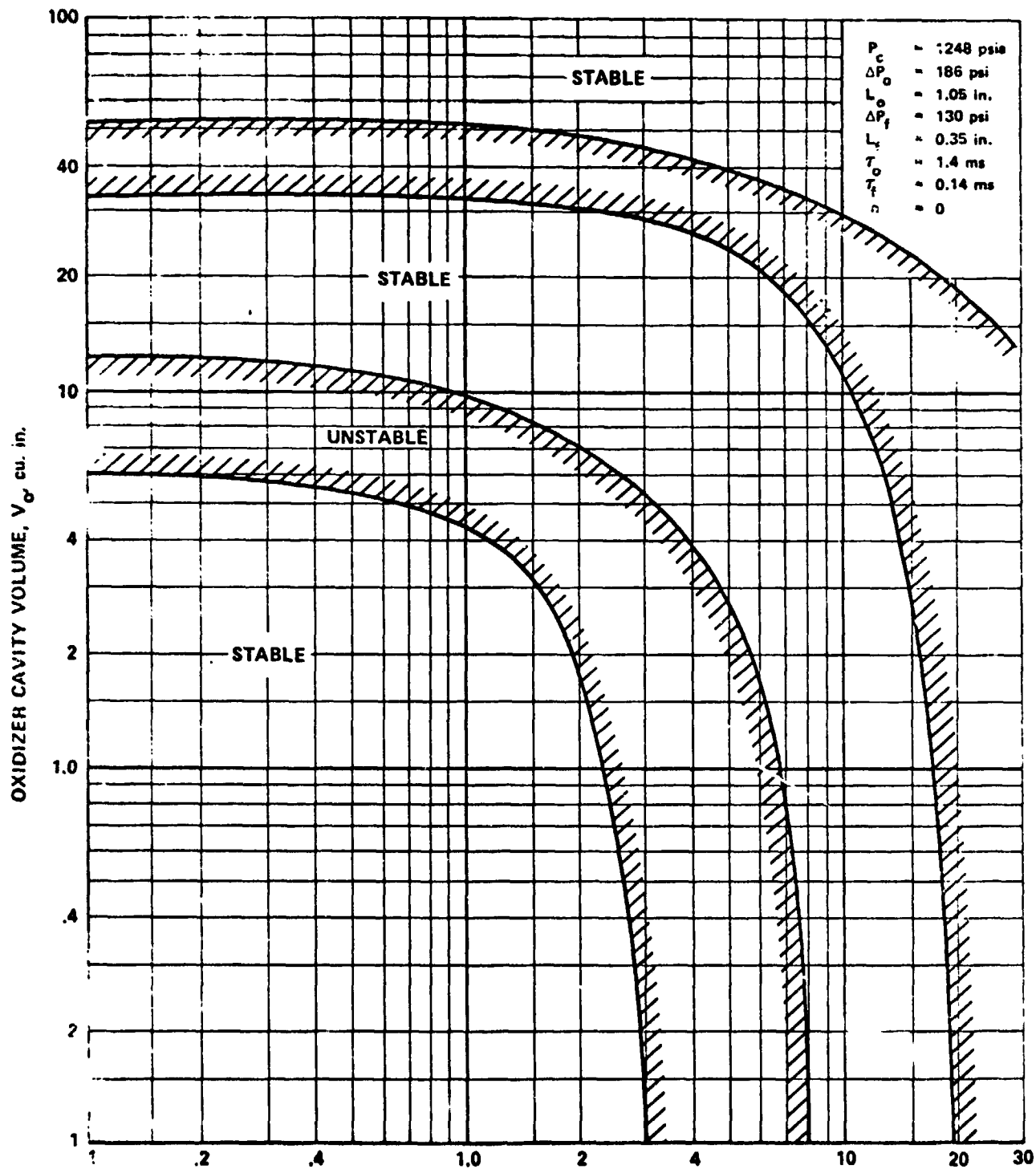


Fig. 6. FUEL CAVITY VOLUME, V_f , cu. in.

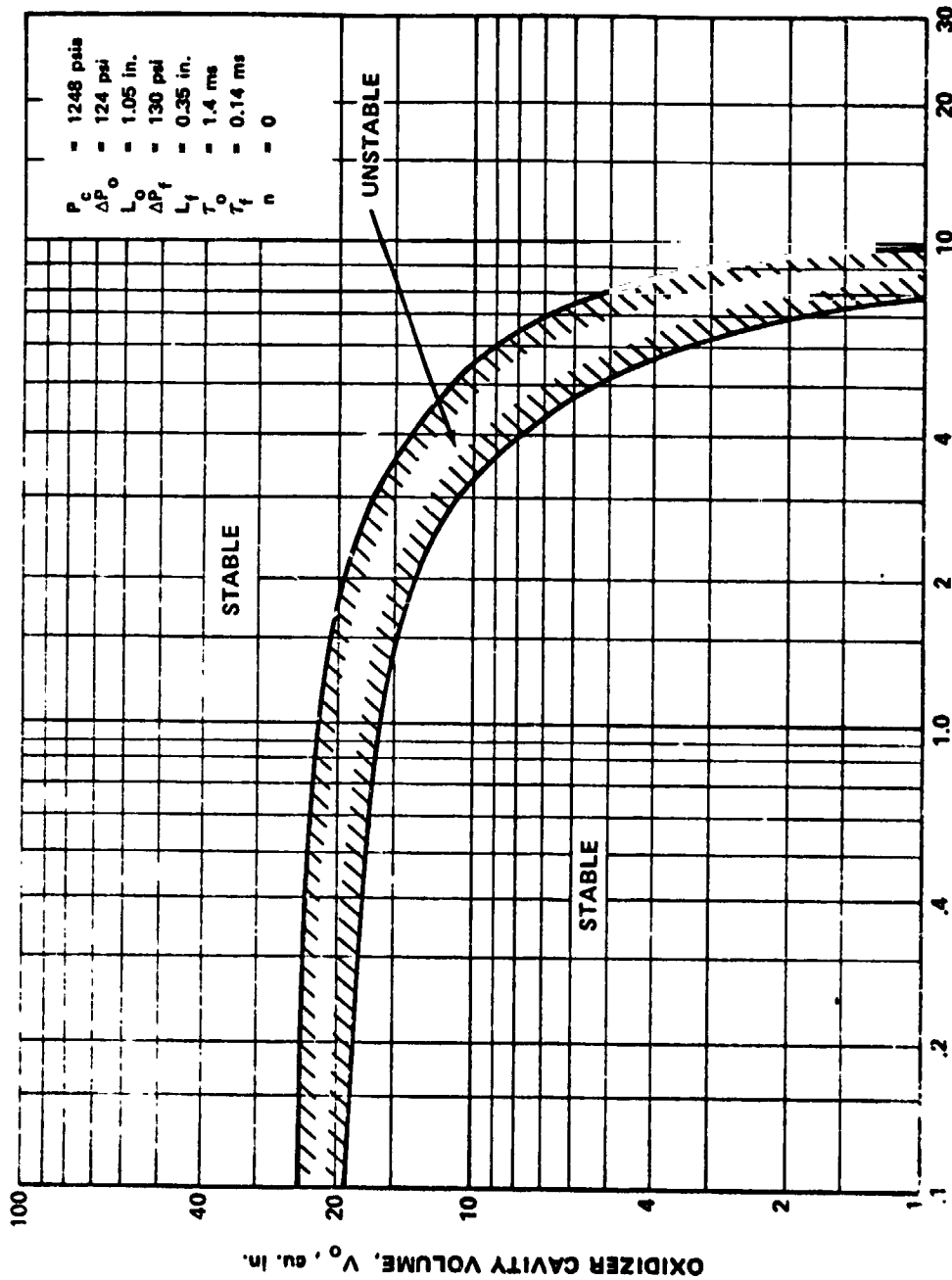


Fig. 7. FUEL CAVITY VOLUME, V_f , cu. in.

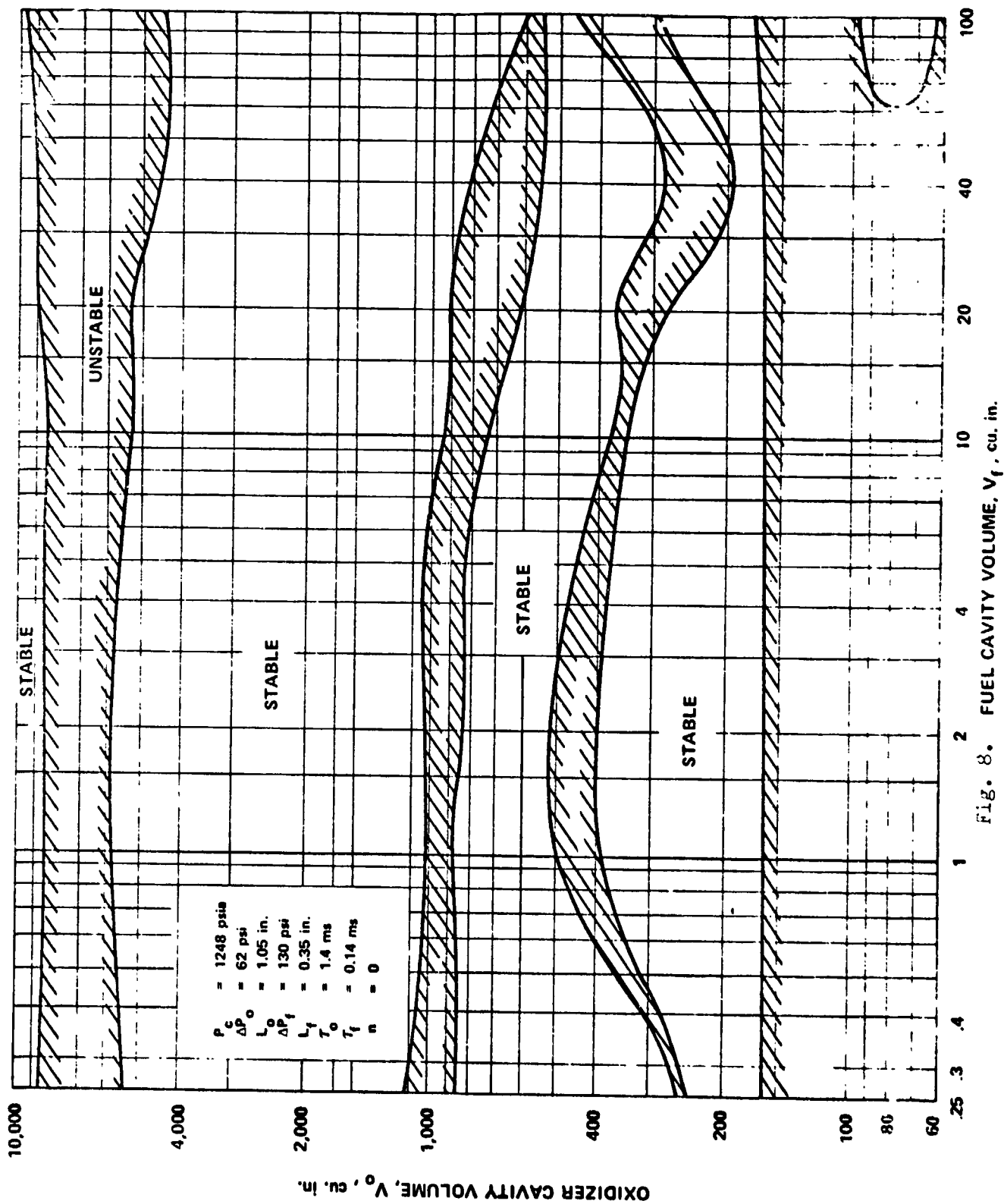


Fig. 8. FUEL CAVITY VOLUME, V_f , cu. in.

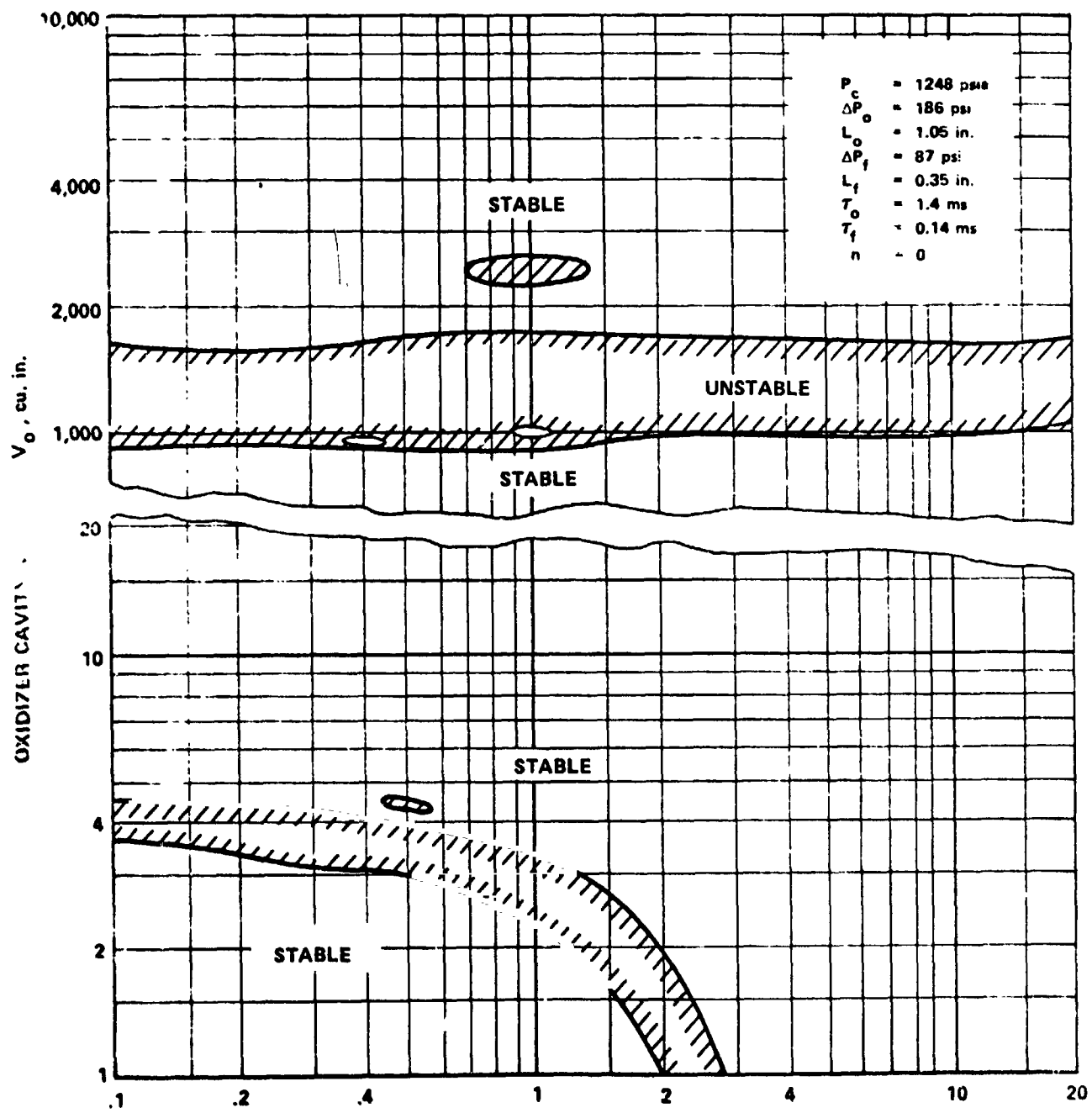


Fig. 9. FUEL CAVITY VOLUME, V_f , cu. in.

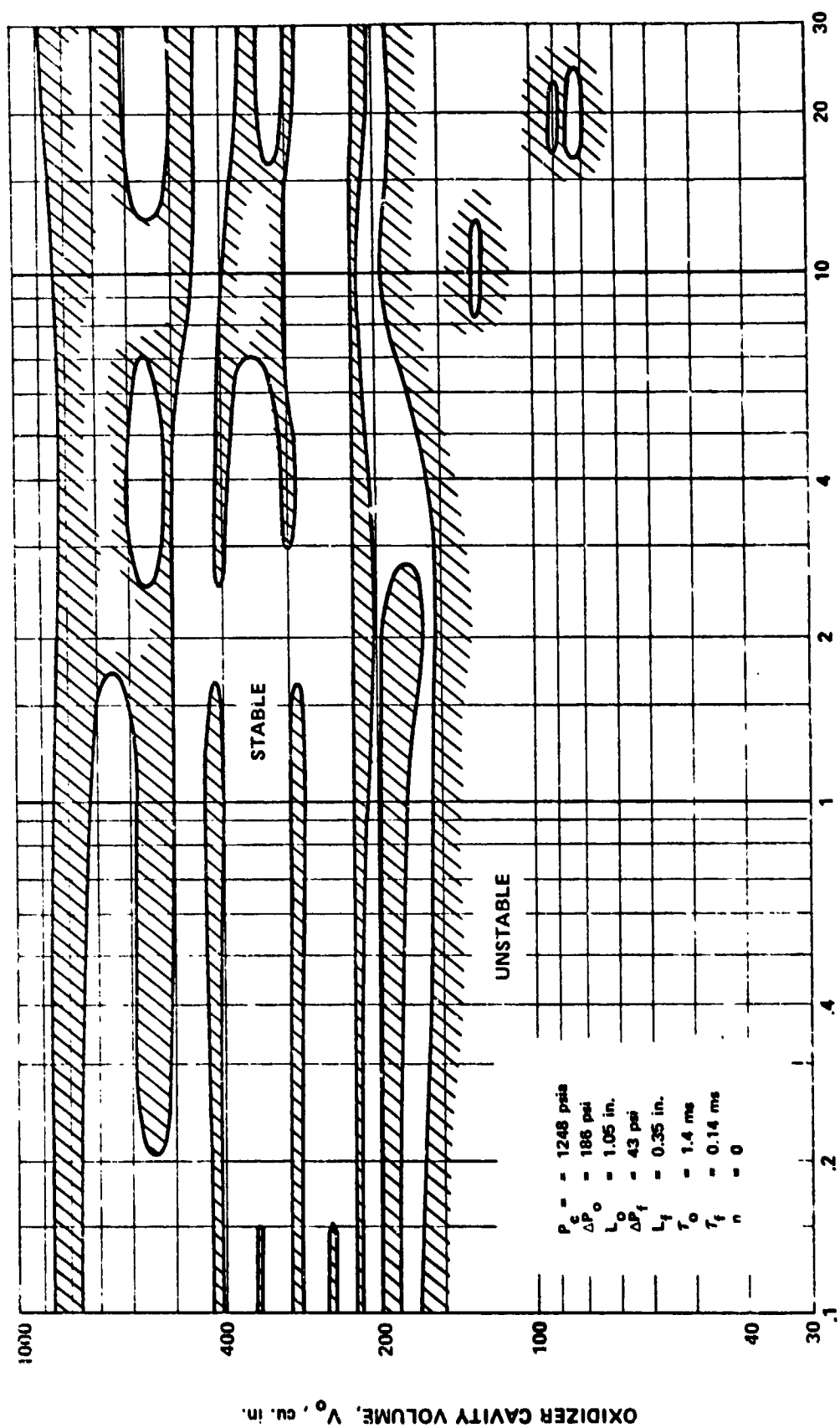


Fig. 10. FUEL CAVITY VOLUME, V_f , cu. in.

CONCLUSIONS

The contribution advanced by the stability model described here is the inclusion of injector inertance and capacitance in addition to resistance in the analysis of bipropellant rocket systems with different time lags. Neutral stability boundaries are shown in terms of these parameters in order to demonstrate their interactions.

This model provides a method of designing a stable system by optimizing the pertinent design variables rather than maximizing the pressure drop and ignoring the others which has been the traditional approach. This model suggests that in some cases stability can be enhanced by reducing pressure drop, and therefore maintaining pressure drop to chamber pressure ratio is not necessarily desirable.

REFERENCES

1. Summerfield, Martain, "A Theory of Unstable Combustion in Liquid Propellant Rocket Systems." J. Am. Rocket Soc., Vol. 21, No. 5, Sept. 1951, pp. 108-114.
2. Gunder, D.F., and Friant, D.R., "Stability of Flow In a Rocket Motor." J. Appl. Mech., Vol. 17, No. 3, Sept. 1950, pp. 327-333.
3. Crocco, Luigi and Cheng, Sin-I. Theory of Combustion Instability in Liquid Propellant Rocket Motors. London: AGARDograph No. 8, Butterworths Sci. Pub., 1956.
4. Hurrell, Herbert G., "Analysis of Injection-Velocity Effects on Rocket Motor Dynamics and Stability." NASA TR R-43, 1959.
5. Wenzel, L.M. and Szuch, J.R., "Analysis of Chugging in Liquid-Bipropellant Rocket Engines Using Propellants with Different Vaporization Rates." NASA TN D-3080, 1965.
6. Tsien, H.S., "The Transfer Functions of Rocket Nozzles". Jl. Amer. Rocket Society 22 (1952) 139, 172. Also pp. 31 of 3.
7. Priem, R.J. and Heidmann, M.F., "Propellant Vaporization as a Design Criterion for Rocket Engine Combustion Chambers." NASA TR R-67, 1960.

Subscript

c

Combustion chamber

o

Oxidizer

f

Fuel

i

Initial

Superscript

n

Indicates dimensionⁿ quantities

LIST OF SYMBOLS

SYMBOL	DEFINITION
D	Differential operator, d/at
E	Dimensionless elasticity parameter
J	Dimensionless inertia parameter
P	Dimensionless pressure drop parameter
p^*	Pressure
p	Dimensionless pressure
M	Molecular weight
V	Volume
T	Temperature
\dot{m}	Mass flow rate
R	Universal gas constant
p	Pressure drop
ρ	Density
X	Compliance
θ_g	Gas residence time
l	Injector orifice length
A	Injector orifice area
s	Laplace transform variable
\dot{w}	Weight flow rate
t	Dimensionless time
n	Interaction index
α	Dimensionless Damping Coefficient
α^*	Damping Coefficient
ω	Dimensionless frequency
ω^*	Frequency
τ	Dimensionless time lag
τ^*	Time lag
$\bar{\tau}$	Dimensionless sensitive time lag
$\bar{\tau}^*$	Sensitive time lag
i	$\sqrt{-1}$

Appendix A

Definition of Parameters

The gas residence time, θ_g is used to nondimensionalize many of the quantities used in the analysis. It is defined as

$$\theta_g = \frac{V_c \rho_c}{\dot{m}} = \frac{V_c \rho_c M_c}{R T_c \dot{m}}$$

The pressure drop parameter, P is defined as

$$P = \frac{\rho_c}{2\Delta\rho}$$

The elasticity parameter, E is defined as

$$E = \frac{2\Delta p p X}{\dot{m} \theta_g}$$

The inertia parameter, J is defined as

$$J = \frac{l \dot{m}}{2\Delta p A \theta_g}$$

The real and imaginary parts of the Laplace transform variable S (α and ω), the oxidizer and fuel combustion delays, and the sensitive time lag are nondimensionalized by the use of the gas residence time, θ_g , namely,

$$\alpha = \alpha^* \theta_g$$

$$\omega = \omega^* \theta_g$$

$$\tau_o = \tau_o^* / \theta_g$$

$$\tau_f = \tau_f^* / \theta_g$$

$$\tau = \tau^* / \theta_g$$

Appendix B

Solution of Equations (61) and (62) by Newton-Raphson Method

For convenience, Equations (61) and (62) are denoted, symbolically and respectively, as

$$f(\alpha, \omega) = 0 \quad (\text{A-1})$$

$$g(\alpha, \omega) = 0 \quad (\text{A-2})$$

By series expansion, Equations (1) and (2) are rewritten

$$\begin{aligned} f(\alpha, \omega) = f(\alpha_{n-1}, \omega_{n-1}) &+ (\alpha - \alpha_{n-1}) \frac{\partial f}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) \\ &+ (\omega - \omega_{n-1}) \frac{\partial f}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) \end{aligned} \quad (\text{A-3})$$

$$\begin{aligned} g(\alpha, \omega) = g(\alpha_{n-1}, \omega_{n-1}) &+ (\alpha - \alpha_{n-1}) \frac{\partial g}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) \\ &+ (\omega - \omega_{n-1}) \frac{\partial g}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) \end{aligned} \quad (\text{A-4})$$

where $\alpha_{n-1}, \omega_{n-1}$ are the values of α and ω at stage of computation $n-1$.

Now if at a subsequent stage n , where $\alpha = \alpha_n$ and $\omega = \omega_n$ the right-hand sides of (3) and (4) vanish, the following can be written immediately.

$$\begin{aligned} \alpha_n \frac{\partial f}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) &+ \omega_n \frac{\partial f}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) = -f(\alpha_{n-1}, \omega_{n-1}) \\ &+ \alpha_{n-1} \frac{\partial f}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) \\ &+ \omega_{n-1} \frac{\partial f}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) \end{aligned} \quad (\text{A-5})$$

$$\begin{aligned} \alpha_n \frac{\partial g}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) &+ \omega_n \frac{\partial g}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) = -g(\alpha_{n-1}, \omega_{n-1}) \\ &+ \alpha_{n-1} \frac{\partial g}{\partial \alpha} (\alpha_{n-1}, \omega_{n-1}) \\ &+ \omega_{n-1} \frac{\partial g}{\partial \omega} (\alpha_{n-1}, \omega_{n-1}) \end{aligned} \quad (\text{A-6})$$

Written in matrix form, (5) and (6) take the form

$$\begin{bmatrix} \frac{\partial f}{\partial \alpha} & \frac{\partial f}{\partial \omega} \\ \frac{\partial g}{\partial \alpha} & \frac{\partial g}{\partial \omega} \end{bmatrix}_{n-1} \begin{bmatrix} \alpha_n \\ \omega_n \end{bmatrix} = - \begin{bmatrix} f \\ g \end{bmatrix}_{n-1} + \begin{bmatrix} \frac{\partial f}{\partial \alpha} & \frac{\partial f}{\partial \omega} \\ \frac{\partial g}{\partial \alpha} & \frac{\partial g}{\partial \omega} \end{bmatrix}_{n-1} \begin{bmatrix} \alpha_{n-1} \\ \omega_{n-1} \end{bmatrix} \quad (\text{A-7})$$

Inversion of (7) yields

$$\begin{bmatrix} \alpha_n \\ \omega_n \end{bmatrix} = - \begin{bmatrix} \frac{\partial f}{\partial \alpha} & \frac{\partial f}{\partial \omega} \\ \frac{\partial g}{\partial \alpha} & \frac{\partial g}{\partial \omega} \end{bmatrix}_{n-1}^{-1} \begin{bmatrix} f \\ g \end{bmatrix}_{n-1} + \begin{bmatrix} \alpha_{n-1} \\ \omega_{n-1} \end{bmatrix} \quad (\text{A-8})$$

$$= \begin{bmatrix} \alpha_{n-1} \\ \omega_{n-1} \end{bmatrix} - \frac{1}{DFG} \begin{bmatrix} \frac{\partial g}{\partial \omega} & -\frac{\partial f}{\partial \omega} \\ -\frac{\partial g}{\partial \alpha} & \frac{\partial f}{\partial \alpha} \end{bmatrix}_{n-1} \begin{bmatrix} f \\ g \end{bmatrix}_{n-1}$$

where

$$DFG = \begin{bmatrix} \frac{\partial f}{\partial \alpha} & \frac{\partial f}{\partial \omega} \\ \frac{\partial g}{\partial \alpha} & \frac{\partial g}{\partial \omega} \end{bmatrix}_{n-1} \quad (\text{A-9})$$

Iteration follows until the roots α and ω are pin-pointed within desired accuracy.

Such a scheme of solving simultaneous equations is known as the Newton-

Raphson method.

A computer program was developed such that for given parameters for a specific rocket engine, for instance,

$$P_0 = 3.36, \quad P_f = 4.82, \quad J_0 = 1.189, \quad E_0 = 0.2018,$$

$$\tau_0 = 5.2692, \quad \tau_f = 0.4, \quad J_f = 0.061, \quad E_f = 0.233,$$

the fundamental roots are obtained

$$a = -0.112$$

$$u = 0.49.$$

Other examples are solved similarly and the results of computation are shown in Figs. i and ii.

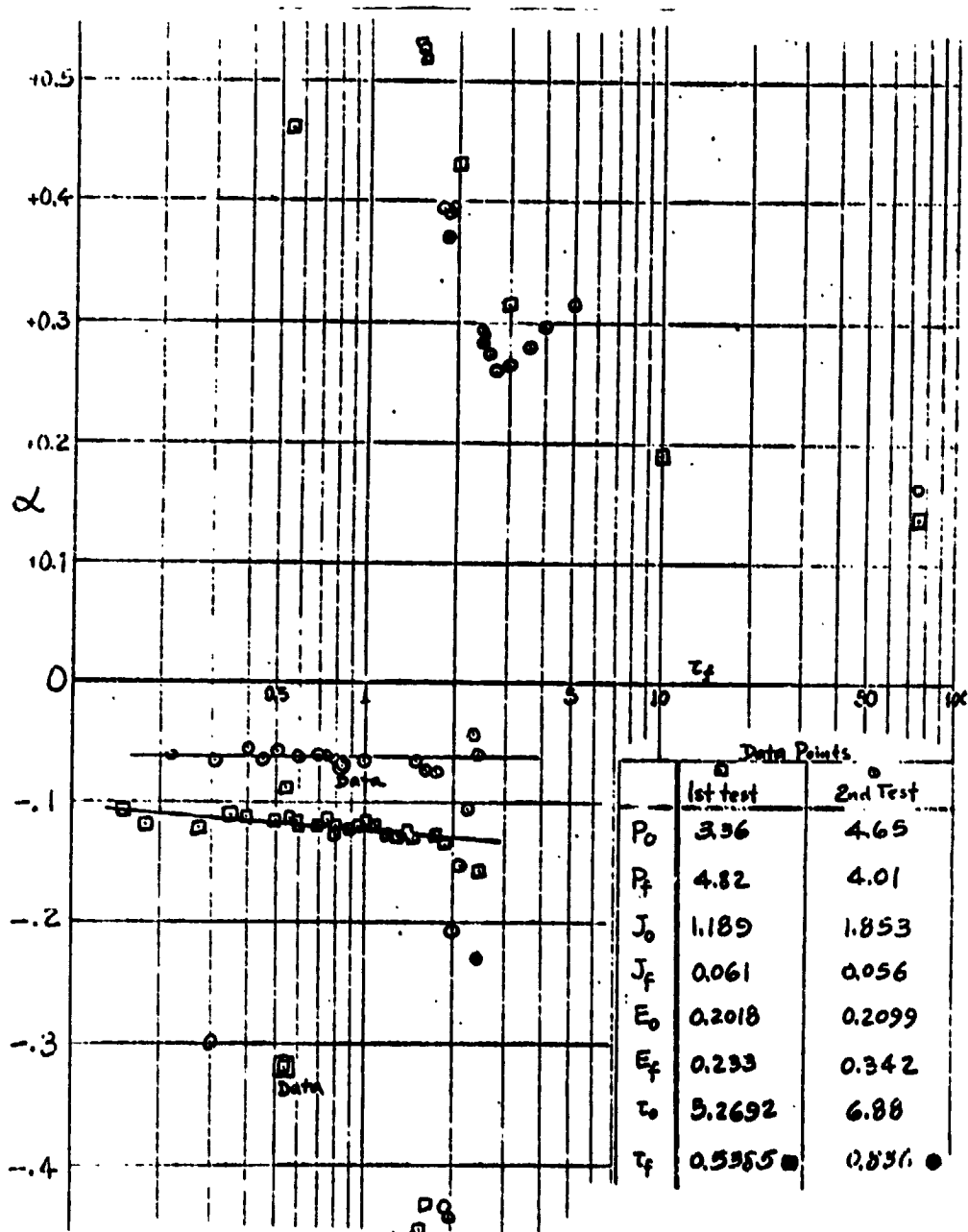


Fig. 1. Survey of α with Different Values of z_f

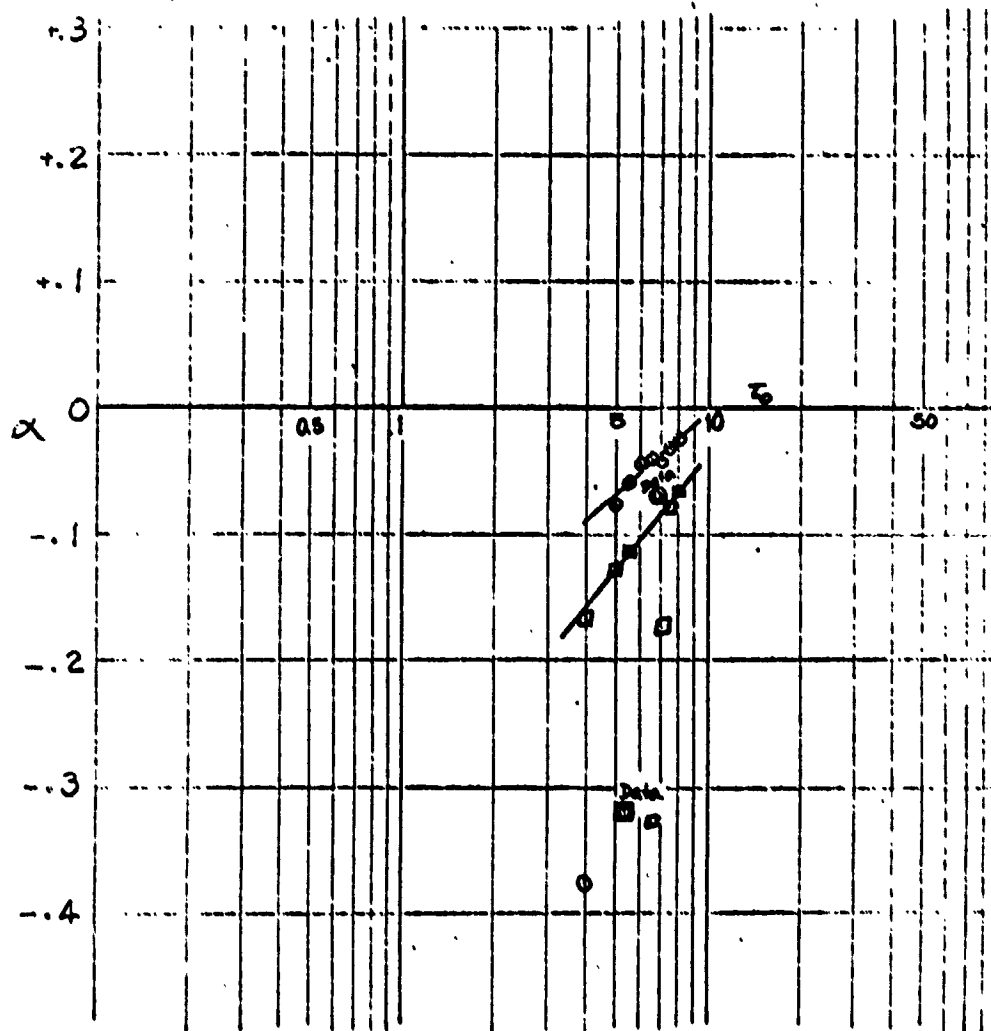


Fig. 11. Survey of α with Different Values of ζ

APPENDIX C

Computer Programs and Sample Calculations

```

$IMH
C RIORDPELLANT
C MARCH 29, 1973
C
C CURVE 3, FIG. 1, PER MARCH 8, DATA FROM RICHMOND
C SEEKING A PLOT OF LQ VERSUS LF
1 N=1
2 READ(5,100) NCASE
3 100 FORMAT(12)
4 1 READ(5,101) RLO,RLF
5 101 FORMAT(2F10.6)
6
7 GC=32.2
8 THG=0.00026
9
10 DPQ=62.
11 ATQ=7.5
12 RMQ=25.4
13
14 DPF=43.3
15 ATF=0.945
16 RMF=4.30
17
18 PQQ=10.08
19 PFF=6.80
20 PJQ=RLQ*RMQ/(24.*DPQ*ATQ*GC*THG)
21 PJF=RLF*RMF/(24.*DPF*ATF*GC*THG)
22 PQQ=0.0721
23 PFF=3.241
24
25 TQ=5.385
26 TF=0.5395
27
28 C THE FOLLOWING CAPDS CONTAIN INFO FOR LATER USE IN VARYING VVO VS VVF
29 C
30 PCC=1248.
31 VVO=12.5
32 WHQD=0.0411
33 VVF=9.5
34 WHFF=0.0002395
35 RKQ=0.01
36 RKF=0.680
37 PQQ=WHQD*RKQ*VVO/(RMQ*THG*(PQQ+0.5))
38 PFF=WHFF*RKF*VVF/(RMF*THG*(PFF+0.5))
39 PQQ=5.04 WAS USED IN CURVE 2, FIG. 1
40 DPQ=124 WAS USED IN CURVE 2, FIG. 1
41 DPF=43.6 WAS USED IN CURVE 2, FIG. 1
42 PQQ=0.1387 WAS USED IN CURVE 2, FIG. 1
43 C END OF LATENT INFORMATION
44
45 AL=0.01
46 W=0.01
47
48 17 FORMAT(1H1)
49 18 WRITE(6,17)
50 18 FORMAT(9X,'LINEAR ENGINE COMBUSTION STABILITY STUDY',/)
51 18 WRITE(6,18)
52 20 FORMAT(10X,'LQ=',E10.4,5X,'LF=',E10.4,/)
53 20 WRITE(6,20) RLO,RLF

```

```

31 19 FORMAT(15X,'ALPHA',11X,'ALPHA 1',15X,'OMEGA',10X,'OMEGA 1',/)
32 WRITE(6,19)
33 5 ET4=1.
34 EVTA=EXP(-T)*AL)
35 EVFA=EXP(-TF*AL)
36 PDC1=COS(T*W)
37 PPS1=SIN(T*W)
38 PDC2=COS(TF*W)
39 PPS2=SIN(TF*W)
40 PPCT=1.
41 PPST=0.
42 AAWH=AL*AL-W*W

C
43 PG=1.+PJ*AL+PJ*PE*AAWH
44 PH=1.+PJF*AL+PJF*PEF*AAWH
45 PK=PJ*W*(1.+2.*PE*AL)
46 PNH=PJF*W*(1.+2.*PEF*AL)
47 PM=PG-PJ*AL
48 PH=PH-PJF*AL
49 PU=2.*PJ*PE*AL*W
50 PV=2.*PJF*PEF*AL*W
51 PA=AL+1.-PM*PN*ETA*PPCT
52 PR=W-PN*ETA*PPST
53 CC=EVOA*PPC1
54 FF=EVOA*PPS1
55 UD=EVFA*PPCF
56 FF=EVFA*PPSF
57 AJ1=PG*PH-PN*PK
58 AJ2=PH*PK+PN*PG
59 AJ5=PM*PH-PU*PN
60 AJ7=PM*PN+PH*PU
61 AJ9=PR*PG-PK*PV
62 AJ11=PR*PK+PG*PV
63 PGA=PJ*W*(1.+2.*PE*AL)
64 PHA=PJF*W*(1.+2.*PEF*AL)
65 PKA=2.*PJ*PE*W
66 PNA=2.*PJF*PEF*W
67 PMA=2.*PJ*PE*AL
68 PQA=2.*PJF*PEF*AL
69 PU=PKA
70 PVA=PNA
71 PAA=1.
72 PBA=0.
73 CCA=-T*EVOA*PPC1
74 FEA=-T*EVOA*PPS1
75 FFA=-TF*EVFA*PPCF
76 FFA=-TF*EVFA*PPSF
77 PGW=-PKA
78 PHW=-PNA
79 PKW=PGA
80 PNW=PHA
81 PMW=PGW
82 PRW=PMW
83 PUW=PMA
84 PVW=PRA
85 PAW=-PHA
86 PBW=PAA
87 CCW=FEA
88 EEW=-CCA
89 DDW=FFA

```

```

200      FFW=-DDA
210      CK1=PG*PHA+PGA*PH-PNN*PKA-PNA*PK
220      CK2=PH*PKW+PHW*PK-PNN*PGW-PNW*PG
230      CK3=PG*PHW+PGW*PH-PNN*PKW-PNW*PK
240      CK4=PH*PKA+PHA*PK-PNN*PGA-PNA*PG
250      CK5=PHA*PH+PH*PHA PU*PNA-PUA*PNN
260      CK6=PHW*PK+PH*PKW-PUN*PNN-PU*PNW
270      CK7=PMA*ONN+PM*PNA+PH*PUA+PHA*PU
280      CK8=PMW*PNN+PM*PNW+PHW*PU+PH*PUW
290      CK9=PR*PG+PR*PGA-PKA*PV-PK*PVA
300      CK10=PDW*PG+PR*PGW-PKW*PV-PK*PVW
310      CK11=PRA*PK+PR*PKA+PGA*PV+PG*PVA
320      CK12=PRW*PK+PR*PKW+PGW*PV+PG*PVW
330      FFF=PA*AJ1-PB*AJ2+PNN*(CC*AJ5+EE*AJ7)+PFF*(DD*AJ9+FF*AJ11)
340      GGG=PR*AJ1+PAA*AJ2+PDN*(CC*AJ7-EE*AJ5)+PFF*(DD*AJ11-FF*AJ9)
350      FFA=PAA*AJ1+PA*CK1-PBA*AJ2-PB*CK4
360      1 +PDN*(CCA*AJ5+CC*CK5+FFA*AJ7+EE*CK7)
370      1 +PFF*(DDA*AJ9+DD*CK9+FFA*AJ11+FF*CK11)
380      C FFF=FFF
390      C
400      FFFW=PAW*AJ1+PA*CK3-PBW*AJ2-PB*CK2
410      1 +PDN*(CCW*AJ5+CC*CK6+FFW*AJ7+EE*CK8)
420      1 +PFF*(DDW*AJ9+DD*CK10+FFW*AJ11+FF*CK12)
430      C FFFW=FFF
440      C
450      GGGG=PBA*AJ1+PB*CK1+PAA*AJ2+PA*CK4
460      1 +PDN*(CCA*AJ7+CC*CK7+FFA*AJ5+EE*CK5)
470      1 +PFF*(DDA*AJ11+DD*CK11+FFA*AJ9+FF*CK9)
480      C GGGG=GGG
490      C
500      GGGW=PRW*AJ1+PR*CK3+PAW*AJ2+PA*CK2
510      1 +PDN*(CCW*AJ7+CC*CK8+FFW*AJ5+EE*CK6)
520      1 +PFF*(DDW*AJ11+DD*CK12+FFW*AJ9+FF*CK10)
530      C GGGW=GGG
540      C
550      DFG=FFFA*G-GG-GGGA*FFF
560      GGG1=(GG-FFF-FFF*GGG)/DFG
570      A1=AL-GG
580      GGG2=(-L-GA*FFF+FFFA*GGG)/DFG
590      W1=W-GGG2
600      C
610      16 FORMAT(10X,2E16.8,5X,2E16.8)
620      WRITE(A,16) AL,A1,W,W1
630      IF(ABS(GGG1).LT.(0.001).AND.ABS(GGG2).LT.(0.001)) GO TO 50
640      AL=A1
650      W=W1
660      GO TO 5
670      50 IF(NCASE-N)52,52,51
680      51 N=N+1
690      GO TO 1
700      52 WRITE(A,17)
710      STOP
720      END

```

SEENTRY

LINEAR ENGINE COMBUSTION STABILITY STUDY

LM=0.1000E 01 LF=0.5000E 00

ALPHA	ALPHA 1	OMEGA	OMEGA 1
0.10000000E-01	0.51556450E 00	0.10000000E-01	0.41743270E-01
0.51556450E 00	-0.49484670E 00	0.41743270E-01	0.69839530E-01
-0.49484670E 00	-0.3099270E 00	0.69839530E-01	0.72969730E-01
-0.3099270E 00	-0.10960670E 00	0.72969730E-01	0.85579990E-01
-0.10960670E 00	0.15632870E 00	0.85579990E-01	0.15707180E 00
0.15632870E 00	0.89898940E-01	0.15707180E 00	0.86538460E 00
0.89898940E-01	0.49778040E 00	0.86538460E 00	0.37635260E 00
0.49778040E 00	-0.27112540E 00	0.37635260E 00	0.31035610E 00
-0.27112540E 00	-0.11232140E 00	0.31035610E 00	0.33325400E 00
-0.11232140E 00	-0.78895090E-02	0.33325400E 00	0.39047810E 00
-0.78895090E-02	-0.11411030E-03	0.39047810E 00	0.43621950E 00
-0.11411030E-03	-0.47754940E-02	0.43621950E 00	0.43635740E 00
-0.47754940E-02	-0.47305670E-02	0.43635740E 00	0.43636080E 00

```

1000
C HPAJPELLANT
C THIS IS CURVE 2, FIG. 4, PER DATA FROM RICHMOND ON MARCH 8, 1973
C SKKING A PLOT OF V0 VERSUS VF
1 N=1
2 P=AD(5,100) NCASE
3 100 FORMAT(12)
4 1 READ(5,101) VV0,VVF
5 101 FORMAT(2F10.4)
6 PN=0.
7 GC=32.2
8 THG=0.00026
9 ATN=0.6
10 ATF=0.945
11 RKN=0.01
12 PKF=0.680
13 RHN=25.4
14 RMF=4.30
15 RHON=0.0411
16 RHFF=0.0002385
17 ONN=124.
18 DPF=46.6
19 PUN=3.36
20 PFF=7.21
21 PJ0=1.189
22 PJF=0.747
23 PEO=RHON*RNK*VV0/(RNO*THG*(P00+0.5))
24 PEF=RHFF*RK*VVF/(RMF*THG*(PFF+0.5))
25 T0=5.385
26 TF=0.5385
27 AL=0.01
28 W=0.01
29 17 FORMAT(1H1)
30 WRITE(6,17)
31 19 FORMAT(9X,'LINEAR ENGINE COMBUSTION STABILITY STUDY',/)
32 WRITE(6,18)
33 20 FORMAT(10X,'V0=' ,E10.4,5X,'VF=' ,E10.4,/)
C V0 AND VF ARE THE OXIDIZER AND FUEL CAVITY VOLUMES, RESPECTIVELY
34 WRITE(6,20) VV0,VVF
35 19 FORMAT(15X,'ALPHA',11X,'ALPHA 1',15X,'OMEGA',10X,'OMEGA 1',/)
36 WRITE(6,19)
37 4
38 ETA=1.
39 EVNA=EXP(-T0*AL)
40 EVFA=EXP(-TF*AL)
41 PPO=COS(T0*W)
42 PPS=SIN(T0*W)
43 PPF=COS(TF*W)
44 PPT=SIN(TF*W)
45 PPS=1.
46 PPT=0.
47 AAW=AL*AL-W*W
48 PG=1.+PJ0*AL+PJ0*PE0*AAW
49 PH=1.+PJF*AL+PJF*PEF*AAW
50 PK=0.01*W*(1.+2.*PE0*AL)
51 PNN=PJF*W*(1.+2.*PEF*AL)
52 PH=PG-PJ0*AL
53 PH=PH-PJF*AL
54 PH=2.*PJ0*PE0*AL*W
55 PV=2.*PJF*PEF*AL*W
56 PA=AL+1.-PN+PN*ETA*PPCT

```

56 DH=H-PN*ETA*PPST
 57 CC=FVDA*PPCQ
 58 EE=FVDA*PPSQ
 59 QD=EVFA*PPCF
 60 FF=EVFA*PPSF
 61 AJ1=PG*PH-PNN*PK
 62 AJ2=PH*PK+PNN*PG
 63 AJ5=PH*PH-PJ*PNN
 64 AJ7=PM*PNN+PH*PU
 65 AJ9=PR*PG-PK*PV
 66 AJ11=PR*PK+PG*PV
 67 DGA=PJI*(1.+2.*PI*AL)
 68 PHA=QJH*(1.+2.*PI*AL)
 69 PKA=2.*PJ*PEQ*W
 70 PNA=2.*PJ*PEF*W
 71 PMA=2.*PJ*PEF*AL
 72 PRA=2.*PJ*PEF*AL
 73 PUJA=PKA
 74 PVA=PNA
 75 PAA=1.
 76 PBA=0.
 77 CCA=-TQ*EVDA*PPCQ
 78 FFA=-TQ*EVDA*PPSQ
 79 QDA=-TF*EVFA*PPCF
 80 FFA=-TF*EVFA*PPSF
 81 DGH=-PKA
 82 PHW=-PNA
 83 PKW=PGA
 84 PNH=PHA
 85 PMW=PCW
 86 PRW=PHW
 87 PIW=PMA
 88 PVW=PRA
 89 PAW=-PRA
 90 PRW=PAA
 91 CCW=FFA
 92 FFH=-CCA
 93 QDW=FFA
 94 FFH=-DDA
 95 CK1=PG*PHA+DGA*PH-PNN*PKA-PNA*PK
 96 CK2=PH*PKW+PHW*PK+PNN*PGW+PNW*PG
 97 CK3=PG*PHW+PGW*PH-PNN*PKW-PNW*PK
 98 CK4=PH*PKA+PHA*PK+PNN*PGA+PNA*PG
 99 CK5=PHA*PH+PH*PHA-PJ*PNA-PUJA*PNN
 100 CK6=PMW*PH+PM*PHW-PUW*PNN-PJ*PNN
 101 CK7=PMW*PNN+PM*PNA+PH*PUA+PHA*PU
 102 CK8=PMW*PNN+PM*PNN+PMW*PU+PH*PUW
 103 CK9=PR*PG+PR*PGA-PK*PV-PK*PVA
 104 CK10=PHW*PG+PR*PGW-PK*PV-PK*PVW
 105 CK11=PHA*PK+PR*PKA+DGA*PV+PG*PVA
 106 CK12=PHW*PK+PR*PKW+PGW*PV+PG*PVW
 107 FFF=PAA*AJ1-PH*AJ2+PQQ*(CC*AJ5+EE*AJ7)+PFF*(DD*AJ9+FF*AJ11)
 108 GGC=PH*AJ1+PA*AJ2+PQQ*(CC*AJ7+EE*AJ5)+PFF*(DD*AJ11+FF*AJ9)
 109 HHA=PAA*AJ1+PA*CK1-PHA*AJ2-PB*CK1
 110 IIA=PQQ*(CCA*AJ5+CC*CK5+FEA*AJ7+EE*CK7)
 111 JJB=PFF*(DDA*AJ9+DD*CK9+FFA*AJ11+FF*CK11)
 KKC=PAW*AJ1+PA*CK3-PRW*AJ2-PB*CK2
 LLD=PQQ*(CCW*AJ5+CC*CK6+FFW*AJ7+FE*CK8)
 MME=PFF*(DDW*AJ9+DD*CK10+FFH*AJ11+FF*CK12)
 NNG=PAA*AJ1+PH*CK1+PAA*AJ2+PA*CK4

```

1      +P.11*(CCA*AJ7+CC*CK7-EEA*AJ5-EE*CK5)
1      +PFF*(DDA*AJ11+DD*CK11-FFA*AJ9-FF*CK9)
112    GGGW=PHW*AJ1+PH*CK3+PAW*AJ2+PA*CK2
1      +P.11*(CCW*AJ7+CC*CK7-FFW*AJ5-EE*CK6)
1      +PFF*(DDW*AJ11+DD*CK12-FFW*AJ9-FF*CK10)
113    DFG=FFFA*GGGW-GGGA*FFW
114    GGG1=(GGGW*FFF-FFW*GGG)/DFG
115    A1=AL-GGG1
116    GGG2=1-GGGA*FFF+FFFA*GGG/DFG
117    W1=W-GGG2
118    16    FORMAT(10X,2E16.9,5X,2E16.9)
119    WRITE(6,16) A1,A1,W1
120    IF (ABS(GGG1).LT.(0.001).AND.ABS(GGG2).LT.(0.001)) GO TO 50
121    AL=A1
122    W=W1
123    GO TO 5
124    50    IF (NCASE-N) 52,52,51
125    51    N=N+1
126    GO TO 1
127    52    WRITE (6,17)
128    STOP
129    END

```

SCNTRY

LINEAR ENGINE COMBUSTION STABILITY STUDY

V=0.1000E 01 VF=0.3500E 00

ALPHA	ALPHA 1	OMEGA	OMEGA 1
0.10000000E-01	0.15797370E 01	0.10000000E-01	0.14378730E 00
0.15797320E 01	0.14339060E 00	0.14378730E 00	0.15136250E 00
0.18144060E 00	-0.31630990E 00	0.15136250E 00	0.14483610E 01
-0.81644400E 00	-0.63559510E 00	0.14483610E 01	0.19983500E 01
-0.41644410E 00	-0.44706470E 00	0.19983500E 01	0.17831720E 01
-0.46506670E 00	-0.31178210E 00	0.17831720E 01	0.19674070E 01
-0.31178210E 00	-0.17792460E 00	0.19674070E 01	0.19521480E 01
-0.29292460E 00	-0.12670480E 00	0.19521480E 01	0.19349530E 01
-0.12670480E 00	-0.11112710E 00	0.19349530E 01	0.19151100E 01
-0.11112710E 00	-0.11099980E 00	0.19151100E 01	0.19026530E 01
-0.11099980E 00	-0.11099990E 00	0.19026530E 01	0.19012750E 01